

Study Guide

GD&T

Application and Interpretation

Sixth Edition



A

Ø.009@|A|B|

Ø.006 A

.003 A

Ф Ø.016(M) A B C

Bruce A. Wilson The author is GDTP certified by ASME

The author is GDTP certified by ASME in accordance with the qualifications of ASME 14.5.2 in the Senior level.



<.008 A

(2) .021 A B C





080 A B C .030 A B C

Based on the ASME Y14.5-2009 standard



Study Guide

GD&T

Application and Interpretation

Sixth Edition

by

Bruce A. Wilson





Publisher.

The Goodheart-Willcox Company, Inc.

Tinley Park, IL www.g-w.com

$\begin{array}{c} \textbf{Copyright} © \textbf{2016} \\ \textbf{by} \\ \textbf{The Goodheart-Willcox Company, Inc.} \end{array}$

All rights reserved. No part of this work may be reproduced, stored, or transmitted in any form or by any electronic or mechanical means, including information storage and retrieval systems, without the prior written permission of The Goodheart-Willcox Company, Inc.

Manufactured in the United States of America.

ISBN 978-1-63126-115-2

6 7 8 9 - 16 - 19

The Goodheart-Willcox Company, Inc. Brand Disclaimer: Brand names, company names, and illustrations for products and services included in this text are provided for educational purposes only and do not represent or imply endorsement or recommendation by the author or the publisher.

The Goodheart-Willcox Company, Inc. Safety Notice: The reader is expressly advised to carefully read, understand, and apply all safety precautions and warnings described in this book or that might also be indicated in undertaking the activities and exercises described herein to minimize risk of personal injury or injury to others. Common sense and good judgment should also be exercised and applied to help avoid all potential hazards. The reader should always refer to the appropriate manufacturer's technical information, directions, and recommendations; then proceed with care to follow specific equipment operating instructions. The reader should understand these notices and cautions are not exhaustive.

The publisher makes no warranty or representation whatsoever, either expressed or implied, including but not limited to equipment, procedures, and applications described or referred to herein, their quality, performance, merchantability, or fitness for a particular purpose. The publisher assumes no responsibility for any changes, errors, or omissions in this book. The publisher specifically disclaims any liability whatsoever, including any direct, indirect, incidental, consequential, special, or exemplary damages resulting, in whole or in part, from the reader's use or reliance upon the information, instructions, procedures, warnings, cautions, applications, or other matter contained in this book. The publisher assumes no responsibility for the activities of the reader.

The Goodheart-Willcox Company, Inc. Internet Disclaimer: The Internet resources and listings in this Goodheart-Willcox Publisher product are provided solely as a convenience to you. These resources and listings were reviewed at the time of publication to provide you with accurate, safe, and appropriate information. Goodheart-Willcox Publisher has no control over the referenced websites and, due to the dynamic nature of the Internet, is not responsible or liable for the content, products, or performance of links to other websites or resources. Goodheart-Willcox Publisher makes no representation, either expressed or implied, regarding the content of these websites, and such references do not constitute an endorsement or recommendation of the information or content presented. It is your responsibility to take all protective measures to guard against inappropriate content, viruses, or other destructive elements.

Introduction

This study guide has been written to supplement the *GD&T*: Application and Interpretation textbook. The review questions and application problems contained in this study guide can be completed on the basis of the information provided by the textbook. Other textbooks may be used, but it is unlikely that any other textbook will provide all the information necessary to answer all the questions or work all the application problems.

The textbook and this study guide used together to provide the information and practice necessary

to gain a strong working knowledge of dimensioning and tolerancing practices.

A majority of the material in the textbook and the study guide requires an understanding of only basic mathematics. Some of the material requires simple algebra operations, such as solving for one unknown value when two known values are provided. Knowledge of print reading or basic drafting techniques will be helpful in understanding the illustrations and completing application problems.

To get the maximum benefit from the textbook and study guide materials, the following study

methods are recommended.

- 1. Read the objectives at the beginning of each chapter of the study guide prior to reading the corresponding chapter in the textbook.
- 2. As you read the textbook chapter, make a list of questions regarding information that is not understood.
- 3. Complete the review questions and application problems after reading the textbook material.
- 4. Cross off the questions from step 2 and 3 as answers are provided during a classroom presentation. Ask the instructor to provide answers if the presentation does not provide all the answers to your questions.
- 5. Correct the answers to your review questions and application problems on the basis of classroom reviews. The corrected materials will be useful for studying for exams.

The objectives at the beginning of each chapter in this study guide define what you should be able to do after studying the textbook, completing outside study activities, attending classroom lectures, and completing study guide review questions and application problems. The level of achievement will depend to a great extent on the amount of time devoted to studying the textbook and study guide materials. Full mastery of dimensioning and tolerancing methods requires studying the fundamentals, then applying them to real industrial applications.

Individuals who put forth the effort to become proficient in dimensioning and tolerancing methods and use that ability to maximize clarity of product design requirements and provide maximum permissible tolerances will be rewarded with the satisfaction of knowing that they are producing the best possible results.

Bruce A. Wilson

Table of Contents

Chapter 1
Introduction to Dimensioning and Tolerancing
Chapter 2 Dimensioning and Tolerancing Symbology
Chapter 3 General Dimensioning Requirements
Chapter 4 Dimension Application and Limits of Size
Chapter 5 Form Tolerances
Chapter 6 Datums and Datum Feature References61
Chapter 7 Orientation Tolerances81
Chapter 8 Position Tolerancing Fundamentals
Chapter 9 Position Tolerancing—Expanded Principles, Symmetry, and Concentricity
Chapter 10 Runout123
Chapter 11 Profile
Chapter 12 Practical Applications and Calculation Methods

Chapter 1

Introduction to Dimensioning and Tolerancing

Name _	chasen	1.	Date 1/13/80	Class	DFT-lal
_				Oldoc	

Reading

Read Chapter 1 of the *GD&T*: *Application and Interpretation* textbook prior to completing the review exercises.

Objectives

A combination of activities is required to achieve the following objectives. Completing the reading assignment and the following review exercises is an important part of achieving the objectives. Familiarization with the objectives prior to completion of the reading assignment and review exercises will make mastery of the objectives easier. After completing the reading assignment and completing the review exercises, you will be able to:

- Explain the importance of accurately specifying dimensions and tolerances.
- ▼ Recall the history and development of dimensioning and tolerancing methods.
- ▼ Explain how teamwork can result in better definition of the dimensions and tolerances shown on a drawing or in a computer-aided design (CAD) file.
- ▼ Recall the job titles of those who should be on the design process team.
- ▼ Recall the dimensioning and tolerancing skills needed for success in design- or production-related occupations.
- ▼ Analyze some possible industrial changes and the impacts of these changes on dimensioning and tolerancing.
- Understand how views are created using orthographic projection.

Review Exercises

Place your answers in the spaces provided. Show all calculations for problems that require mathematical solutions.

Multiple Choice

1. The wavelength of a specific color of light is used in determining the length of one ____.

A. foot

B. yard C. meter

D. kilometer

Name Choose
True/False
10. True or False? The current standard specifies that all measurements must be in inches.
11. True or False? The designer should work independent of others to achiev an optimum design.
12. True or False? The symbol for inches must be applied to all values less than one inch.
13 True or False? Disagreement about drawing requirements can occur whe nonstandard dimensioning methods are used.
14 True or False? Interpretation of a drawing is the ability to determine part requirements from what is shown when the drawing complies with standards.
:; True or False? A projection symbol should be included on orthographic drawings to indicate whether the views are created using first or third angle projection.
Fill in the Blank
16. A(n) is an ancient unit of measurement based on the distance across a finger. We lect on live 17 may be used to establish or show relationships between features in adjacent orthographic views.
adjacent orthographic views.
Short Answer
18. Why is it important to have an accurate distance standard? Veg is Standards in not ACC water distance standard measure. Not possible to specify Accurate demension
19. Give one reason why nonstandard symbols are generally avoided. Only Standard Symbol. Ove hard defined meaning non Standard Symbol ove not defined
20. Show a note that should be placed on a drawing that primarily has inch dimensions.
Demension are Shawing in inches Unless indicated

Chapter 2

Dimensioning and Tolerancing Symbology

Name	V WOKV)	Date	Class
Reading			
Read Cha exercises.	pter 2 of the GD&T	: Application and Interpretation textbook	c prior to completing the review
Objectives	6		
Familiarization will make mass review exercis ▼ Identify an ▼ Identify an ▼ Complete a	n with the objective stery of the objectives, you will be able d draw general din d draw tolerancing feature control frasic dimensions and del.	riew exercises is an important part of a es prior to completion of the reading as es easier. After completing the reading to: nensioning symbols and show their ge symbols and show their general appli me using the correct order of segments define means for indicating a basic di	ssignment and review exercises assignment and completing the eneral applications. Ications. In the frame.
		aces provided. Accurately complete ar uire mathematical solutions.	ny required sketches. Show all
Multiple C	hoice		
		A value shown is a reference value. A. in brackets B. underlined C. with an arc above it D. in parentheses The origin symbol is	ae.
		A. applied to one end of all dimensio	ons

B. applied to both ends of some dimensions

C. applied in place of one arrowhead

D. never used

Name		
Fried	13.	True or False? A diameter symbol is placed in front of the tolerance value in all feature control frames.
	14.	True or False? A datum feature symbol, in an orthographic view, may be applied on either side of an extension line without affecting the meaning of the symbol.
	15.	True of False? Symbols are required to be sized proportional to the feature to which the symbols are applied.
	16.	True or False? Tolerance symbols are generally shaped to give an indication of the required control.
	. 17.	True or False? Abbreviations and words are typically used in notes lists, but symbols may be used in notes.
	18.	True of False) All feature control frames must show a material condition modifier following the tolerance value.
Fill in the Blank		
(C) yill r	19.	Using symbols the number of words that are placed on a drawing.
four		There are different form tolerance symbols.
Sw face	21.	Feature control frames specify tolerances to be applied to or features of size.
	22.	Any tolerance applied to a thread and shown in a feature control frame is assumed to apply to the diameter of the thread unless indicated otherwise.
Christian January	23.	A(n) may be used to indicate that all dimensions are basic.
basic	24.	A(n) dimension can be indicated by drawing a rectangle around the dimension value.
KFS	25.	The abbreviation for <i>regardless of feature size</i> is
Short Answer		
26. The letter X may be u	sed a	s a symbol. What are the two possible uses of the symbol X?
177.47	No	of time and in

27.	Explain how each of the meanings for the symbol X is indicated.
28.	How is the symbol is Protionally side to Character
29.	If a drawing is being produced by hand, what is one method of ensuring that symbols are quickly drawn and correctly sized? When Ein and the symbols are quickly drawn and correctly sized?
30.	List the two types of profile tolerance symbols. The names of the symbols must be given.
31.	Describe the total runout symbol.
32.	What is the order in which datums are referenced?
33.	List the three datum target types. ine j Point) Area

12 GD&T: Application and Interpretation Study Guide

Name Magen

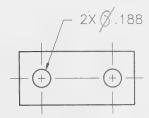
Application Problems

.)

All application problems are to be completed using correct dimensioning techniques. Show any required calculations.

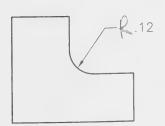
34. Show the diameter symbol in the correct location on each of the diameter dimensions.



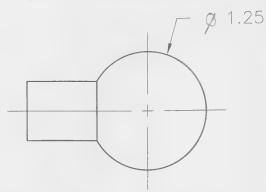


35. Properly show the radius symbol on each of the radius dimensions.

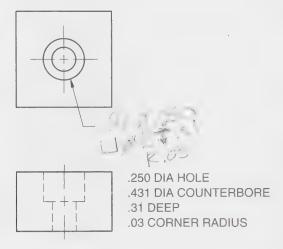




- 14
- 36. Show the spherical diameter symbol on the given dimension.



37. Use symbols to complete the hole and counterbore specification. Correctly position the specification near the leader.



38. Use symbols to complete the hole and countersink specification. Correctly position the specification near the leader.



Name Choosen

39. Label each compartment of the feature control frame.



40. Identify each of the given symbols.

A.	Ø	Diaméter.
В.		17. 11 L 11.
C.	\\\·	CounterSink
D.	$\overline{\vee}$	Veep
E.	A	Eature Feature
F. :3		(),),)(
G.		Straighthess
Н.		
T		

J.	_	Angularity
K.	//	3-1-(1), (1), (1)
L.		profile of a sur
M.	A	Ku
N.	ZA	Total Runout
O.	M	Min Min Comment
P.		least material condition (LMC)
0		The ingression

NOTES	

16 GD&T: Application and Interpretation Study Guide

Chapter 3

General Dimensioning Requirements

Name	U V VSEV	5	Date	Class
Read	ing			
Re		GD&T: Application and	d Interpretation textbook p	prior to completing the review
Objec	ctives			
assignn Familia will ma review App Desc and Utili App Cite	nent and the follow rization with the ol- ke mastery of the of exercises, you will oly general dimension cribe and apply gen- polar coordinate di- ize preferred dimen- oly general and spec-	ing review exercises is objectives prior to compositives easier. After obe able to: oning methods using the training systemsions.	s an important part of act pletion of the reading assi completing the reading a the correct line types, lette stems including chain, ba wide clear part requirements.	ignment and review exercises assignment and completing the ering sizes, and arrowhead form aseline, rectangular coordinate,
mathen	nce your answers in natical solutions. ple Choice	the spaces provided.	Show all calculations for	problems that require
	A	1. Extension lines to provide a vis A015" B062" C125"		from the dimensioned featur

D. .188"

Name	UND. 1		
		t E	Adjacent dimension values are normally to make them easier to read. A. offset B. lined up C. avoided D. None of the above.
		A A	A(n) view sometimes requires that one end of a dimension apply to a hidden feature. A. profile B. auxiliary C) full section D. half section
.}		E G	dimensioning is applying dimensions in such a manner as to result in more than one means of defining the dimension and tolerance on a feature. A. Double B. Duplicate C. Ordinate D. Third angle
			A dimension value placed indicates the value is for reference only. A. between quotation marks inside a rectangle between parentheses D. between brackets
		r F	When the maximum shaft size is equal to the minimum hole size, the mating parts have a zero A. transition B. material condition tolerance allowance
True/Fa	alse		
	,	15.	True or False? Size dimensions define the location of features.
	+	n	True or False? The unidirectional dimensioning system usually requires more space for vertical dimensions than does the aligned dimensioning ystem.
			rue or False? Regardless of the drawing scale, the drawing must show he dimension values to be produced.
	4		rue or False? Visualizing the geometric shapes in a part can help letermine what dimensions are needed.
			<i>True or False?</i> The view in which a feature is dimensioned may be selected trandom.

	_ 20.\	True or False? Dimensioning between views is a good practice that makes it easier to relate dimensions to two views.
7. J. 20 cm.	_ 21.	True of False? Dimensions to hidden features are common since many holes are shown with hidden lines.
	_ 22.	True or False? When possible, all dimensions should be placed on a view in which the dimensioned features are seen in true size.
	_ 23.	True or False? General notes provide information that applies to the entire drawing.
7	_ 24.	True or False? Notes must be shown on the drawing sheets that contain the views of the part.
Fill in the Blank		
dot	_ 25.	A leader line has an arrowhead on end.
1 PISTON	_ 26.	The recommended minimum distance from an object to the first dimension line is
· · · · · · · · · · · · · · · · · · ·	_ 27.	Notes are connected to features using a
	_ 28.	The recommended length-to-width ratio for an arrowhead is
(1/2) 0 /W	_ 29.	The dimensioning system has values aligned with the dimension lines.
(-0100	_ 30.	Rectangular coordinate dimensioning without dimension lines places dimension values at the ends of lines.
Consta	_ 31.	Polar dimensions include a distance and
1000=	_ 32.	A(n) used to replace one of the arrowheads on a dimension line indicates the origin for the dimension.
1.00	_ 33.	A(n) dimension can be indicated by drawing a rectangle around the number.
CONTRO	_ 34.	A feature frame contains a tolerance that is attached to a feature.
Short Answer		
35. When may a leader		e broken?
		Janueles 1:2 Maris 10

GD&T: Application and Interpretation Study Guide

20

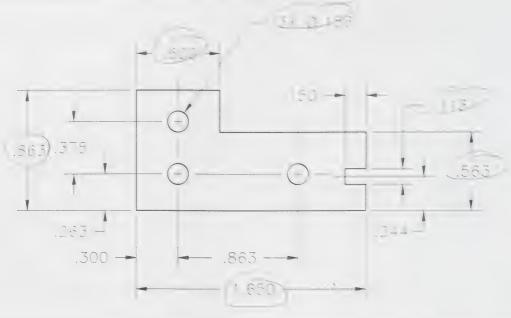
Nam	e Masur III
36.	List two of the possible arrangements for arrowheads and dimension values in relationship to the extension lines.
37.	Why are horizontal and vertical leader lines avoided?
	100000 1 21 212 112 1 10 10 10 10 10 10 10 10 10 10 10 10 1
38.	Describe an advantage of using unidirectional dimensioning over aligned dimensioning in orthographic views.
:3	
39.	When is it necessary to show the unit of measurement for a dimension?
	1010/ 12" mad n (n Simple out)
40.	Why are larger dimensions typically placed outside smaller dimensions? Common C
	CAUCIA TO THE CAUCIA CARREST CONTRACTOR OF THE CAUCIA CARREST CARREST CONTRACTOR OF THE CAUCIA CARREST CONTRACTOR OF THE C
	1/exame a
41.	Where may section lines be broken to make dimension application in a section view more clear?
	MA MON PLAN 1:21/ " 10 10 CISMI SIMEN

Application Problems

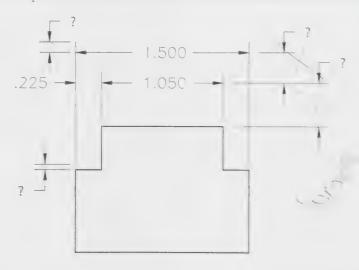
All application problems are to be completed using correct dimensioning techniques. Show any required calculations.

- 42. Show the symbol for each of the following:
 - A. Maximum material condition
 - B. Least material condition

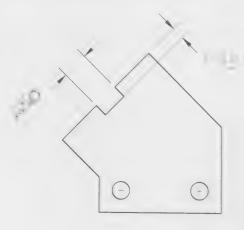
43. Circle the dimension value for each of the size dimensions.



44. In place of each of the question marks, indicate the recommended value for dimensioning.

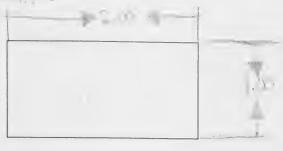


45. Apply dimension values to the shown slot using unidirectional dimensions. The slot is .250" wide and .125" deep.

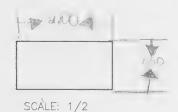


Name

46. A full scale and half scale drawing of the same rectangular part are given. Dimension both of the drawings. The actual size of the rectangle is $2.00'' \times 1.00''$.



SCALE: 1/1



47. What are the maximum and minimum permissible horizontal dimensions between points A and F on a part produced to the given drawing? Assume no form or orientation variation exists to complicate the problem.

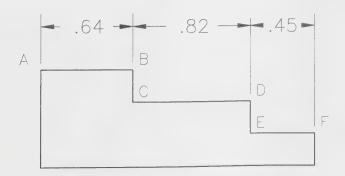
Maximum Maximum

Minimum

TOLERANCES:

$$.XX = \pm .02$$

 $.XXX = \pm .005$



- 24
- 48. What are the maximum and minimum permissible horizontal dimensions between points C and D on a part produced to the given drawing? Assume no form or orientation variation exists to complicate the problem.
 - Maximum

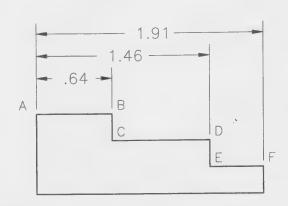
Minimum



TOLERANCES:

$$.XX = \pm .02$$

 $.XXX = \pm .005$



- 49. What is the specified size for hole B1 and what is the allowable size variation?
 - Specified size
 - 6 Allowable size variation

What is the coordinate location for hole B1?

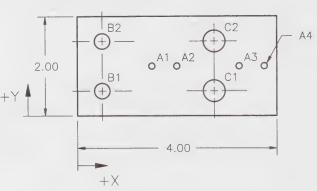
- 4,00 X Coordinate location
- 2.00 Y Coordinate location

What is the specified size for hole A2 and what is the allowable size variation?

- Specified size
- _____ Allowable size variation

What is the coordinate location for hole A2?

- X Coordinate location
- Y Coordinate location

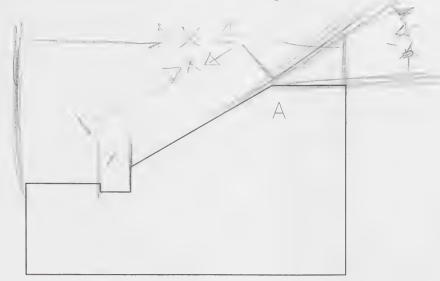


DRILL TABLE

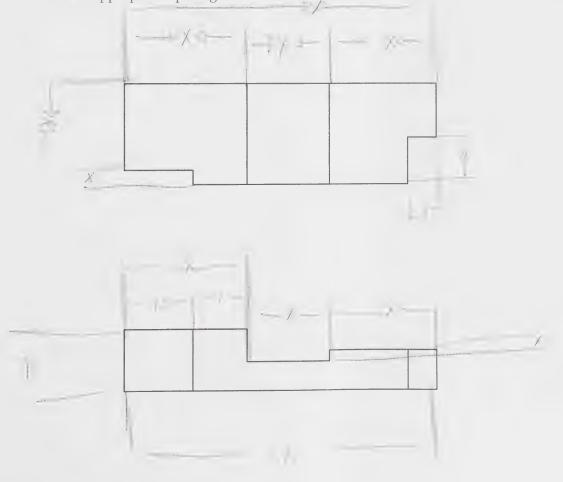
URILL TABLE						
SYMBOL	LOCATION		CIZE	TOI		
JIWIBOL	+X	+Y	SIZE	TOL		
A1	1.50	1.00	.125	+.005 000		
A2	2.00	1.00				
А3	3.25	1.00				
A4	3.75	1.00				
B1	.50	.50	.312	+.005 000		
B2	1.00	1.50				
C1	2.75	.50	.438	+.006		
C2	2.75	1.50	.+56			

Name WW Liv

50. Locate vertex A for the inclined surface and dimension the angle.

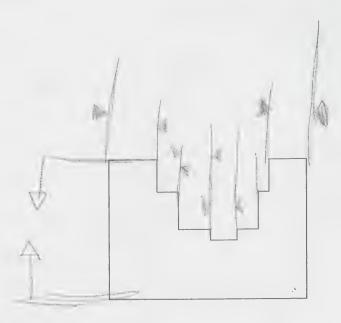


51. Apply dimensions to the given part. Be certain to apply dimensions where the feature profiles are best shown. Use appropriate spacing between dimensions.

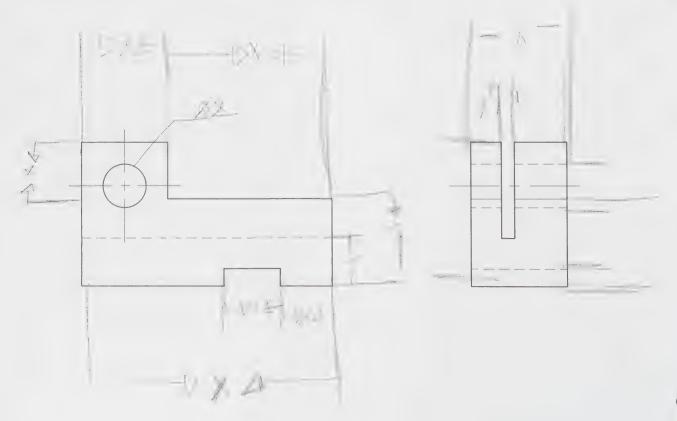


26

52. Dimension all features.

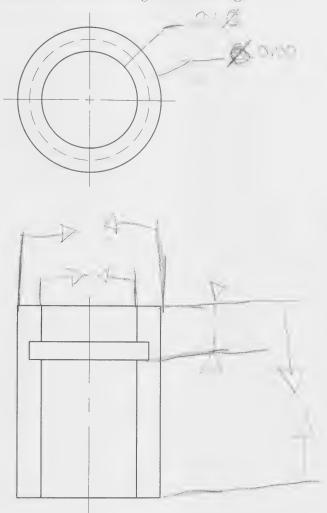


53. Dimension the depth for each slot. Also dimension the location of the hole.

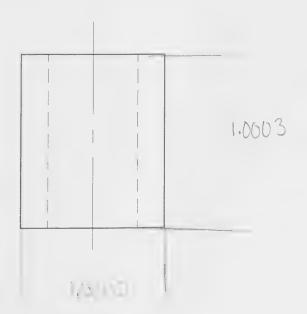


Name _ Challe

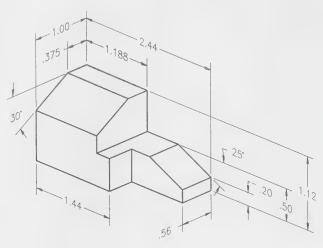
54. Dimension the given section view and add section lining (crosshatching).

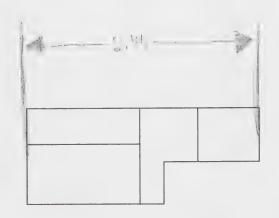


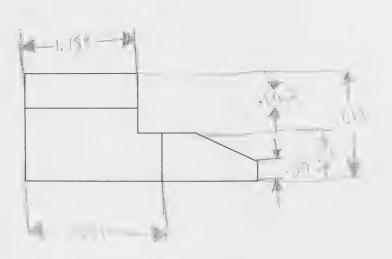
55. Apply 1.0003" and 1.0000" limits of size to the outside diameter.

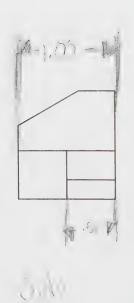


56. Apply the needed dimensions on the orthographic views.





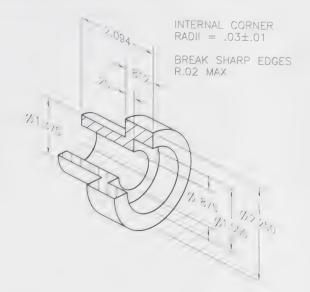


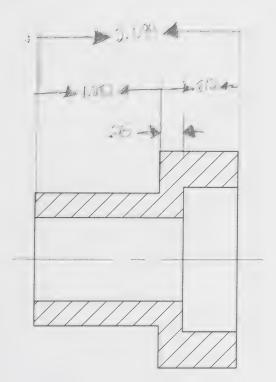


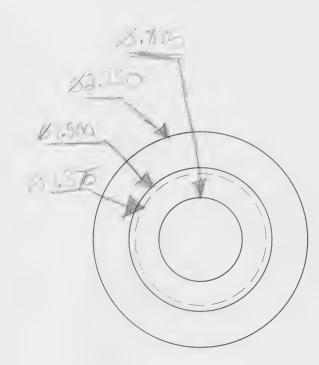


Name Crash

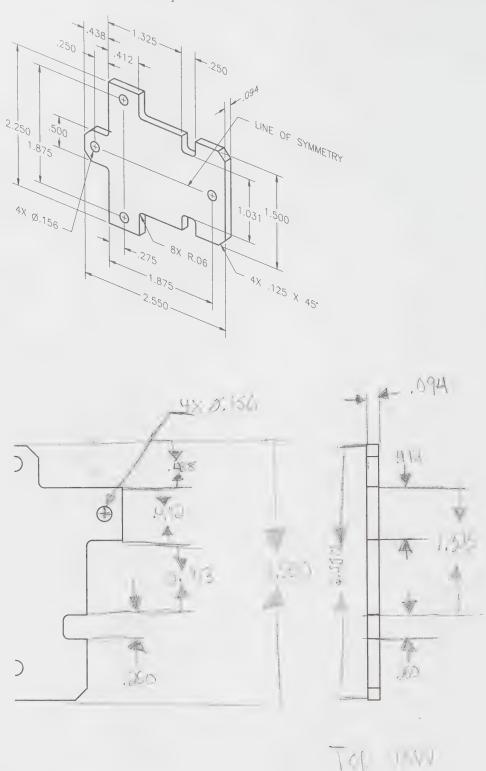
57. Apply the needed dimensions on the orthographic views.







58. Apply the needed dimensions on the views provided.



Chapter 4

Dimension Application and Limits of Size

Name	Date	Class				
Reading						
Read Chapter 4 of the GD&T: Application exercises.	and Interpretation textbool	k prior to completing the review				
Objectives						
A combination of activities is required to assignment and the following review exercises. Familiarization with the objectives prior to conwill make mastery of the objectives easier. After review exercises, you will be able to: Clearly apply dimensions by complying with apply dimensions to any of the geometric apply climensions to any of the geometric apply climits of fit and described to a calculate and apply limits of size for matinal explain Rule #1 and Rule #2 of the ASME Approvide examples of the effects that dimental Complete a surface texture specification with Review Exercises	s is an important part of a mpletion of the reading as er completing the reading at the stated general dimenshapes commonly found ribe the general condition features. (14.5-2009 standard. sions and tolerances have	achieving the objectives. ssignment and review exercises g assignment and completing the nensioning guidelines. on mechanical parts. n created by each category.				
Place your answers in the spaces provided mathematical solutions. Multiple Choice 1. An angle in a drawn perper A. untolerate basic	nn orthographic view is as ndicular to one another.	or problems that require				

D. No assumption permitted.

GD&T: Application and Interpretation Study Guide

Name	
<u> </u>	 The letter R in a radius dimension is shown as a to the dimension value. A. prefix B. suffix Either A or B. Neither A nor B.
	 Although no break is required, extension lines may be broken where they cross A. extension lines B. dimension lines C. object lines D. arrowheads
.,	 The minimum allowable bend radius for a sheet metal part is affected by the A. type of material B. hardness condition of the material C. material thickness All of the above.
(13. A bend radius that is too small can result in that weaken the part. A. ridges B. sharp edges C. cracks D. None of the above.
	 The maximum limit of size is placed the minimum limit of size when shown in a dimension. A. below B. above C. to the right of D. to the left of
	15. When using the system, the limits of size for the shaft are calculated to fit the hole. basic tolerancing position tolerancing basic hole basic shaft
	A. RC B. LC LT D. FN
	 17. Which of the following classes of fit is most likely to result in a clearance condition? A. LT1 LT6 C. LN2 D. FN4

Name		
	31.	<i>True or False?</i> Fabrication accuracy capabilities and methods do not generally need to be considered when applying dimensions or calculating tolerances.
	32.	<i>True or False?</i> The lifecycle costs for mated assemblies can be higher than for interchangeable assemblies.
Fill in the Blank		
to 17/9, 6	33.	The diameter and dimension must be given for a cylindrical part.
- 21. 11	34.	A diameter dimension line applied on a circular view is oriented to pass through the of the dimensioned feature.
	35.	The abbreviation for counterbore is
<u>diff</u> ?(36.	A countersink hole specification includes a hole diameter, countersink, and countersink angle.
3 00	37.	What is the equivalent decimal degree value for 25°30'?
45	38.	Chamfers made at a(n) angle may be dimensioned with a note.
)/:-	39.	The leader for a radius dimension extends through the arc
	40.	Limit dimensions specify the and acceptable dimension values.
a ensin	41.	When using the basic system for calculation of size limits, one size limit for the shaft is the basic size.
1/ MAIN I WAY I	42.	is the direction of tool marks, scratches, or the grain may be specified as part of a surface texture specification.
Short Answer		
43. Explain how a single	view	can be dimensioned to completely define a cylindrical part.
1//		

44.	What is the effect of using very small size tolerances?
	of what wanter of Suga Size toper
	-1, 11-
45.	If a pattern of holes is repeated several times on a drawing, why would a removed view be used to define the hole locations within the pattern?
	Section 1997 to the section of the s
46.	Define counterbore and list one application of a counterbore.
	Complex 15 W I Server I Proyect of the Server I Proyect Only 10 Proyec
	18 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	Arrive on the form
47.	How deep must a spotface be made if no depth dimension is shown?
	Le pthe Sufficient to Create a Vilat Sura
48	How is a centerline identified as a line of symmetry?
10.	Trow is a certained as a line of synthetry.
	Carry La Company
49	What are four pieces of information that must be included in a thread specification?
17.	fair Pieces Threed is nominal size, threed per
	iron, Thread form, and thread class.
50.	How can an exception to Rule #1 be specified?
	11 / 1CD-1

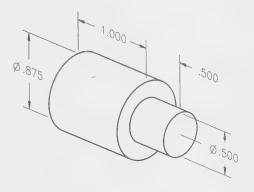
36

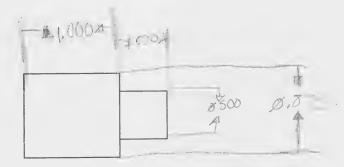
Application Problems

- .;

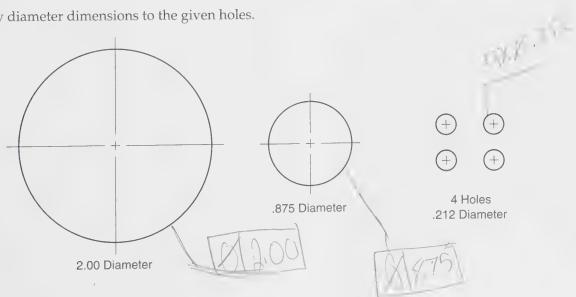
All application problems are to be completed using correct dimensioning technique. Show any required calculations.

51. On the orthographic view provided below, apply all dimensions necessary to completely define the

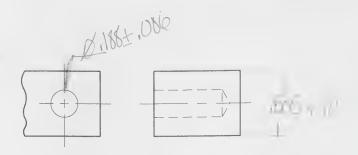




52. Apply diameter dimensions to the given holes.

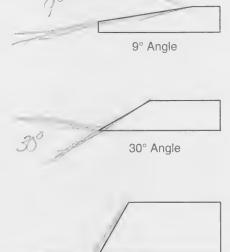


53. Apply a hole specification to the given hole using the given information. Use symbology.



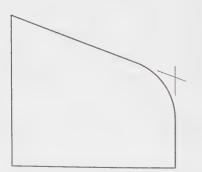
Hole diameter: .188 +.006 - .003 Depth: .500±.010

54. Dimension each of the following angles.



55. Completely dimension each part, estimating dimension values. The arc on one of the parts must be located by dimensioning the tangents. The arc on the other part must be located by dimensioning the arc center. Do not double dimension any feature.

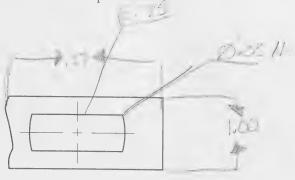
60° Angle



Name Masel

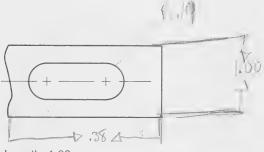
. 3

56. Dimension each slot using the dimension values provided.



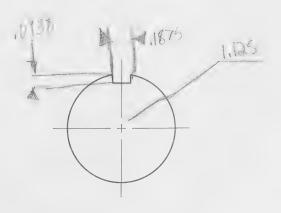
Length: 1.00 Width: .38 Radius: .75

Corner Radius: .03 MAX

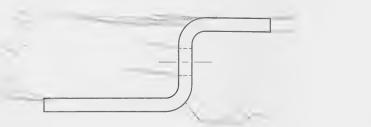


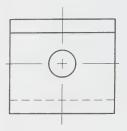
Length: 1.00 Width: .38 Radius: .19

57. Dimension the shaft diameter and the keyseat. Use the dimension information provided.



Shaft diameter: 1.125 Keyseat width: .1875 Keyseat depth: .0938 58. Completely dimension the sheet metal part. Estimate dimension values.





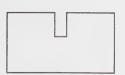
59. Dimension the slot width on each of the given drawings. Use limit dimensions on the indicated part and plus or minus tolerances on the other part. Determine dimension values from the shown information.



Limit dimension Slot width: .125

.125± 20% Plus tolerance: .005

Minus tolerance: .002

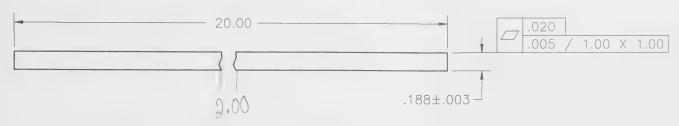


Plus or minus tolerance

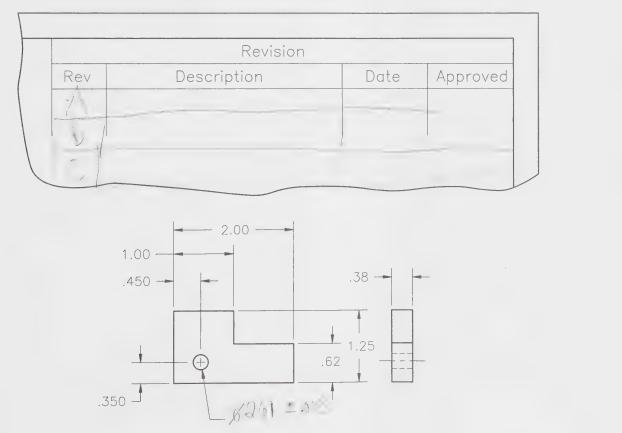
Slot width: .125 Plus tolerance: .005 Minus tolerance: .002 Name ____

.;

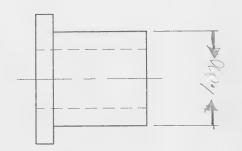
60. Add the necessary information to the drawing to permit exception to the requirements of Rule #1 for the thickness dimension.

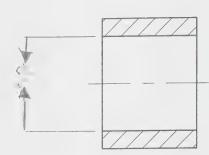


61. Complete the given drawing by entering the information for a revision. The indicated hole was previously dimensioned as a .250" diameter. It is now to be a .261" diameter with a .006" plus tolerance and .003" minus tolerance. Also complete the revision block.

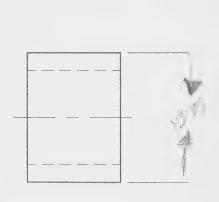


62. Calculate limits of size and apply dimensions for the shown parts. Show all calculations. (See Figure 4-47 of the textbook.) Use tolerance tables in ASME B4.1 or Machinery's Handbook. Apply the dimensions using limit dimensions.





Basic hole system Basic size: 1.000 Class of fit: LC3



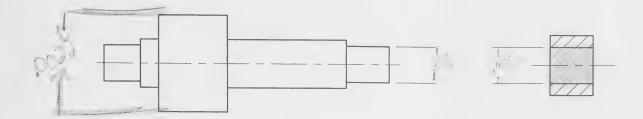


Basic hole system Basic size: 1.375 Class of fit: FN4

Name () Name

. ;

63. Calculate limits of size for the shaft and hole. Show all calculations. Split the allowable tolerance evenly between the two parts.



Basic hole system Basic size: .375 Allowance: .0004

Maximum clearance: .0022

64. Complete a surface texture specification that permits a roughness of 125 microinches with a roughness width cutoff value of .03". No lay direction is required.



65. Complete a surface texture specification that permits a minimum roughness of 63 microinches and a maximum roughness of 250 microinches with a roughness width cutoff of .100". No additional control is needed.



NOTES

Chapter 5

Form Tolerances

Name	Chasen		Date	Class	
Reading					
Re		GD&T: Application ar	ad Interpretation textbook	c prior to completing the review	
Obje	ctives				
assignr Familia will ma review	ment and the following rization with the objects and the objects are symbols for formulate a feature control of the extension of the extension of the extension of the extension and calculate virillaring and calculate virillaring and the extension of the extension of the extension of the extension and calculate virillaring to the extension of the ex	ng review exercises in ectives prior to completives easier. After a able to: In tolerances. In control established ances to control surfolerances. It al condition for a real a surface and show an integer	is an important part of a pletion of the reading as completing the reading a form tolerance and produce elements of a derive an interpretation of the e and show an interpretation of a circular preserved.	ectives. Completing the reading achieving the objectives. ssignment and review exercises assignment and completing the operly apply material condition d median line and show the at has a form tolerance applied to inflatness tolerance zone. ation of the flatness tolerance zone. Indicity tolerance zone.	
calcula	tions for problems th	he spaces provided. at require mathema	Accurately complete ar tical solutions.	ny required sketches. Show all	
Wulti	ple Choice	 Allowable variable specified by six A. position B. orientation 	ze and tolerances.	n individual feature may be	

location

D. form

Name	(\ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \	
	F	10. Parts subject to are not controlled by Rule #1. A free state variation B. damage C. mass production D. None of the above.
		11. Perfect form at MMC is not a requirement when a straightness tolerance is applied to define allowable variation of A a flat surface B. the derived median line of a cylinder curface elements on a cylinder D. None of the above.
	12	12. A specified derived median line straightness tolerance on a shaft A. also establishes a direct control of surface straightness has no direct effect on surface straightness C. must be specified in a special manner to also establish a tolerance fo the surface straightness D. None of the above.
ij		13. If exception to Rule #1 is allowed on a feature, then a must be applied on that feature. A. small size tolerance form tolerance C. surface finish specification D. None of the above.
	· · · · · · · · · · · · · · · · · · ·	14. A straightness tolerance used to specify allowable derived median line variation for a cylinder must include A. an MMC modifier B. no modifier C. a diameter symbol D. None of the above.
	2	 Departure from MMC does not result in any change in the allowable form tolerance if the tolerance is specified to apply at A. MMC B. RFS C. LMC D. All of the above.
		 The virtual condition of a hole is calculated by the MMC size and straightness tolerance value. A. finding the difference between adding C. multiplying D. None of the above.

Name		
	26.	<i>True or False?</i> A form tolerance specification that is applied to one flat surface will also be applicable to any surface that is parallel to the toleranced surface.
	27.	True or False? Stock materials, such as sheet and plate, must meet the requirements of Rule #1.
	28.	True or False? Straightness tolerances are never used to specify derived median line straightness for a shaft.
	_ 29.	<i>True or False?</i> A straightness tolerance may be used to establish allowable variation for surface elements on a cone.
	30.	<i>True or False?</i> A derived median line straightness tolerance may be larger than the size tolerance.
	_ 31.	<i>True or False?</i> Exception to the perfect form boundary requirements created by the size limits is never permitted regardless of the form tolerance values.
.;	32.	True or False? Functional gages may be used to inspect parts that have tolerances specified with the MMC modifier.
	33.	True or False? Unit length tolerances for derived median line straightness must be specified with a unit length of one inch.
	_ 34.	<i>True or False?</i> Flatness tolerance feature control frames never include datum references to establish orientation requirements for the toleranced features.
	35.	<i>True or False?</i> A flatness tolerance that is attached to one surface establishes a requirement for that surface plus any other parallel surface.
	36.	True or False? Flatness of a derived median plane may only be specified by applying two flatness tolerances, one on each of the two surfaces that establish the derived median plane.
	37.	<i>True or False?</i> Circularity tolerances may be applied to any feature with a circular cross section.
	38.	True or False? Rule #1 in ASME Y14.5 defines what is often referred to as the envelope principle.
Fill in the Blank		
Mayaray	_ 39.	The four form tolerances are straightness, flatness, circularity, and
feature.	_ 40.	All form tolerances are specified in a control frame.
Subendole	_ 41.	Unless shown otherwise, the material condition modifier on a form tolerance is assumed to be

50	GD&T: Application and I	nterpre	etation Study Guide
	3,0	42.	A hole specification of .375" ±.005" diameter results in a perfect form boundary of diameter.
	bolles	43.	A tolerance specifies how close to perfectly straight a feature must be made.
	10, 07	44.	A straightness tolerance applied to a feature of size is assumed to apply with the modifier unless shown otherwise.
	381	45.	The virtual condition for a .375" ±.003" diameter shaft with a derived median line straightness tolerance of .007" diameter is
	101-	46.	The MMC modifier indicates that the specified tolerance value may as the toleranced feature departs from the MMC size.
	trom	47.	Additional tolerance gained due to specification of the MMC modifier and departure of a feature from MMC is known as tolerance.
	rave	48.	Two parallel bound the tolerance zone for a flatness tolerance.
	Cyane	49.	Two circles are the tolerance zone boundaries for a circularity tolerance.
	torm	50.	tolerances simultaneously establish requirements for circularity and straightness of cylindrical surfaces.
Sho	ort Answer		
		ions o	on an individual feature specified to a value less than the size tolerance?
	ner	MIM	Marchael Cocycle 111. 1 m. J.
$\frac{v}{-q} = \frac{v}{a}$			18 36 36 36 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
52.	List the form toleran	ce cat	regories.
	21111	(%)	regories.
53.	to what type of featu	ire?	ifier is applicable to the tolerance value when a form tolerance is applied
	14/1.7		CONTRACTOR ANTER LOND 40 OFFICE
			in the lenner
54.			ween a surface and a feature of size.
	<u>a-111</u> a) -	ice is t'e size hinst orginal

Name (\\\)

55. Define maximum material condition.

MMC is when Novimum permissible am.

56. When all features on a part are at MMC, why is it possible for two adjacent features of size to be at an imperfect angle to one another?

Desferct to imperfect andle totall weather.

57. Describe free state variation.

-40 policies in it is a section of the section of t

58. Describe how an exception to Rule #1 may be specified for a single feature.

Copen is form Bou:

59. Define virtual condition.

Viringi (come of the waster of

60. Explain the difference between a straightness tolerance specified on a flat surface and a flatness tolerance applied to the same surface.

to Straightness tolerane and floatness

tolerane and floatness

tolerane and floatness

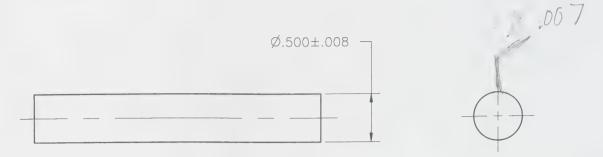
Application Problems

All application problems are to be completed using correct dimensioning techniques. Show any required calculations.

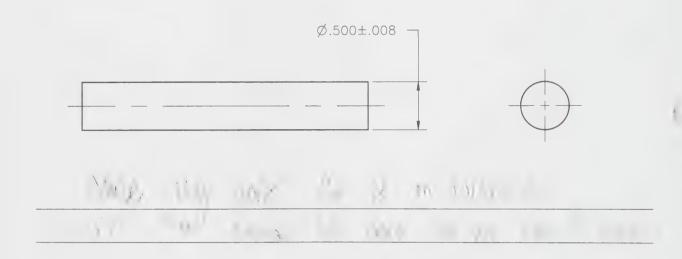
61. If the bottom surface of a part produced to the given drawing is perfectly flat, what is the maximum possible flatness variation on the top surface?

447 .437±.010

62. Apply a straightness tolerance of .007" to control the straightness of surface elements on the given shaft.

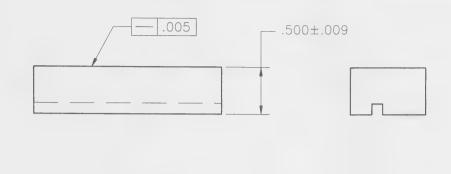


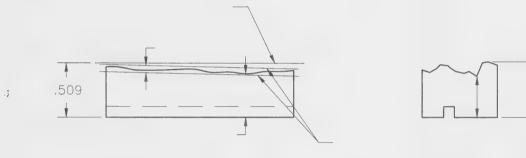
63. Apply a straightness tolerance of .007" at MMC to specify a derived median line (axis) straightness on the given shaft or explain why it cannot be done.



Name Majle Lu

64. Complete the interpretation drawing for the specified tolerances. Add any required tolerance zone boundaries, dimensions, or notes needed to complete the interpretation.

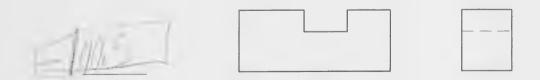




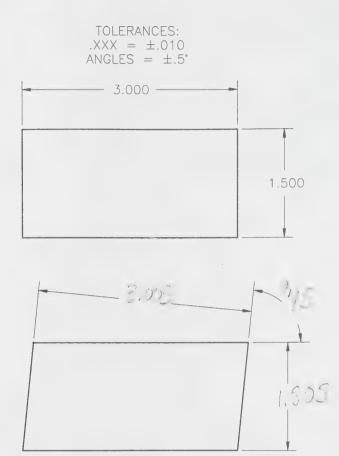
One cross section lengthwise through the part

One cross section across the part

65. Number the surfaces on the given part and enter the total number of surfaces on the blank provided.



66. Apply measurement values on the given part to illustrate the worst-case scenario that is allowed when both features are at MMC.



67. Show two methods of applying a straightness tolerance of .008" on the bottom surface of the given view. Also show a thickness dimension of .750" \pm .015".



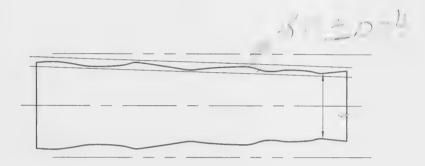
68. Complete a straightness tolerance specification of .008" diameter at MMC.



Name ____

69. Complete the interpretation drawing for the specified tolerances. Add any required tolerance zone boundaries, dimensions, or notes needed to complete the interpretation.



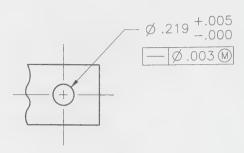


70. Complete the interpretation drawing for the specified tolerances. Add any required tolerance zone boundaries, dimensions, or notes needed to complete the interpretation.





- 56
- 71. What is the virtual condition for the hole?



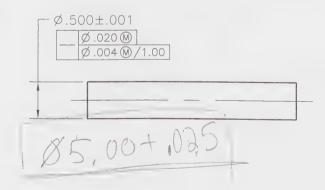
72. Apply a straightness tolerance specification that results in a virtual condition of .216" diameter.



73. Apply a straightness tolerance specification to achieve overall length derived median line straightness of .015" diameter at MMC and unit length derived median line straightness of .005" diameter at MMC per 1.00" of length.

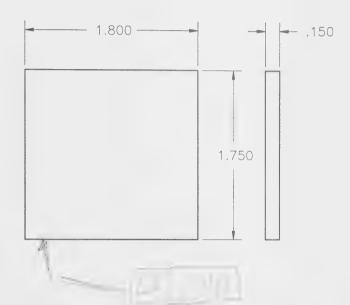


74. Sketch a gage to check the unit length specification in the given figure. Apply dimensions to show the theoretical dimensions for a perfect gage. Do not apply gage tolerances.

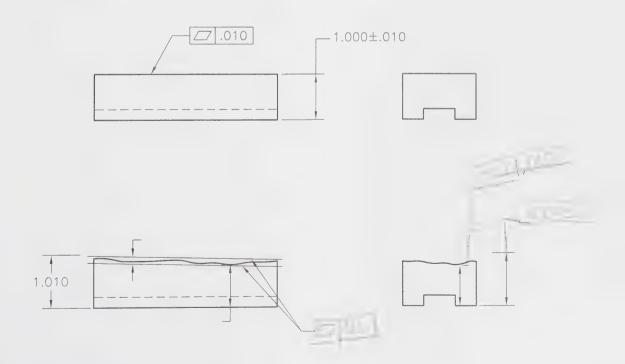


Name _____

75. Show two methods of applying a flatness tolerance of .010" on one of the large surfaces on the part in the following illustration.



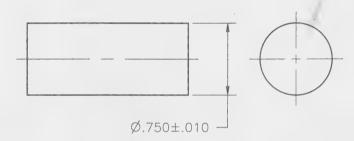
76. Complete the interpretation drawing for the specified tolerances. Add any required tolerance zone boundaries, dimensions, or notes needed to complete the interpretation.



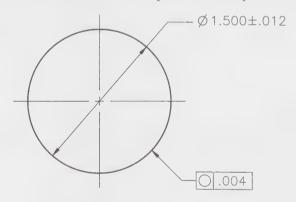
77. Draw a feature control frame that establishes an overall flatness tolerance of .020" and a unit area flatness of .009" per square inch.

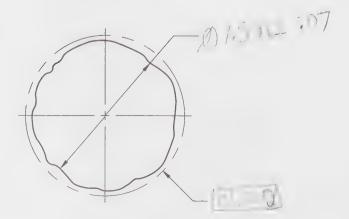


78. Apply a circularity tolerance that permits .010" surface variation when measured radially from a perfect circumscribing circle.



79. Complete the interpretation drawing for the specified tolerances. Add any required tolerance zone boundaries, dimensions, or notes needed to complete the interpretation.

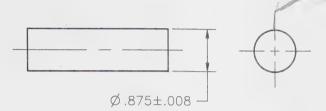




Name ______

. 3

80. Apply a tolerance specification that requires surface conditions to fall within two concentric cylinders separated by .005".



81. A shaft is produced at a diameter of .559". The specified size is .562" ±.004" and a derived median line straightness tolerance of .003" diameter at MMC is specified. What is the allowable straightness variation on the produced part?

60	GD&T: Application and Interpretation Study Guide
NO	TES
140	

Chapter 6

Datums and Datum Feature References

Name Vigiliary	Date	Class	
Danding			

Reading

Read Chapter 6 of the GD&T: Application and Interpretation textbook prior to completing the review exercises.

Objectives

A combination of activities is required to achieve the following objectives. Completing the reading assignment and the following review exercises is an important part of achieving the objectives. Familiarization with the objectives prior to completion of the reading assignment and review exercises will make mastery of the objectives easier. After completing the reading assignment and completing the review exercises, you will be able to:

- Define the difference between a theoretically perfect datum and a datum feature.
- ▼ Explain how to create a datum reference frame through references made in a feature control frame.
- ▼ Utilize all methods for identifying datum features, including the use of target points, lines, and areas.
- ▼ Make datum feature references in a feature control frame using the correct order of precedence.
- ▼ Explain how a datum reference frame may be simulated when three mutually perpendicular surfaces are referenced as datum features.
- ▼ Use material boundary modifiers on datum feature references and explain the significance of the modifiers.
- ▼ Identify the degrees of freedom constrained by each referenced datum feature in a datum reference frame.

Review Exercises

Place your answers in the spaces provided. Accurately complete any required sketches. Show all calculations for problems that require mathematical solutions.

Multiple Choice

1. Datum feature references may be contained in a ________
A. datum reference frame

B. feature control frame

C. datum system

D. machine part

Name _		
	3	 10. A leader extending from a datum target symbol to a datum target indicates the target is on the far side of the object. A solid B. dashed C. phantom D. None of the above.
		11. Single point contact at a target point can be achieved with a A. side of a round dowel chuck or collet spherical-ended tool post All of the above.
	70	12. An end view of a is shown with the same symbol as a target point A, target line B. target area C. datum surface D. None of the above.
		shape shape round square rectangular Any of the above.
	<u>.</u>	 datum reference frame(s) is/are created if one feature control frame references datum A primary, B secondary, and C tertiary; and another feature control frame references datum B primary, C secondary, and A tertiary. A. One B. Two C. Three D. All of the above.
	/\	 15. A flat surface on a part will stabilize on point(s) or more when set on a surface plate. A. one B. two C. three D. None of the above.
		 Datum is a means of approximating the theoretical location of the datums. A. referencing B. identification C. targeting D. simulation

Name	
26.	True or False? The letter used for a primary datum feature reference mus
1	precede the letter in the alphabet used for a secondary datum reference.
27.	<i>True or False?</i> Using implied datums is permitted since this practice saves time.
28.	<i>True or False?</i> A datum target point shown on a drawing indicates that the target location is intended to make point contact with the tooling.
29.	<i>True or False?</i> Contact with a datum target line on a flat surface may be achieved by contacting the side of a dowel pin.
30.	<i>True or False?</i> The perimeter of a target area must always be shown with a phantom line.
31.	<i>True or False?</i> Datum precedence shown in a feature control frame affect how the datum features are used to establish a datum reference frame.
.; 32.	<i>True or False?</i> A secondary datum feature that is produced with an angular variation relative to the primary datum feature causes the datum reference frame to be distorted.
33.	<i>True or False?</i> The minimum number of points on a flat surface that mus make contact to establish a secondary datum plane is two.
34.	<i>True or False?</i> A datum feature triangle should not be attached to a dimension line.
35.	<i>True or False?</i> Before a means of datum simulation can be determined, it is necessary to know the order of precedence of all datums and the material boundary modifier applicable to each reference.
36.	<i>True or False?</i> A datum feature cannot be referenced as a primary datum in one specification and as a secondary datum in another specification.
37.	<i>True or False?</i> Multiple (compound) datum feature references separated by a dash create a requirement to use the identified features to establish one datum.
38.	<i>True or False?</i> ASME Y14.5 specifies that datum feature symbols should not be shown on centerlines.
39.	<i>True or False?</i> Datum targets are permitted on cylindrical features such a holes and shafts.
40.	<i>True or False?</i> More than three datum targets may be placed on a single datum feature.
41.	True or False? It is a poor practice to combine datum target areas and datum target points on the same datum feature.

Fill in the Blank		
-	42.	A datum reference frame made up of three mutually perpendicular planes may be established by referencing datum feature(s) that are flat surfaces.
1	43.	A feature symbol is used to identify a surface or feature of size as a datum feature.
100	_ 44.	A datum is established by a flat surface that is identified as a datum feature.
	45.	A(n) line (<i>type of line</i>) is normally used to show the perimeter of a datum target area.
	46.	A primary datum feature establishes location of the first plane in the datum frame.
<u> </u>	47.	points are required to define a plane.
400	48.	The secondary datum plane in a datum reference frame must be oriented to the primary plane.
-5	49.	flat surfaces must be referenced to establish three planes in a datum reference frame.
-5	50.	The diameter of a round target area may be shown in the half of the datum target symbol.
	51.	The order of datum shown in a feature control frame must be considered when defining datum targets on a drawing.
+	52.	If a primary datum plane is established by a flat surface, holes must be referenced as datum features to completely establish and clock the datum reference frame.
10	53.	Surfaces that lie in more than one plane are called datum surfaces when they are used to establish one datum plane.
	54.	If a primary datum feature reference is to a datum feature of size and the reference includes the MMB modifier, then the datum simulator size is equal to the of the datum feature.
	55.	If a secondary datum feature reference is to a datum feature of size and the reference includes the MMB modifier, then the datum simulator size is equal to the of the datum feature.

Nan	ne
Short Answer	
56.	What is the difference between a datum feature and a datum?
57.	List two types of tolerance specifications that typically include datum feature references.
58.	State one reason why it is preferable to measure from a datum reference frame rather than from datum features.
	.;
59.	Explain why it is ambiguous to place a datum feature symbol on the centerline of a counterbored hole.
60.	Describe two of the methods for applying a datum feature symbol to indicate that a flat surface is a datum feature.
61.	List the three types of datum targets.

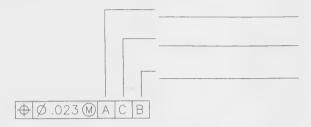
2.	Explain why at least three target points are needed on a surface that is referenced as a primary datum feature.
3.	List one factor that should be considered when determining the size of a datum target area and explain why the factor should be considered.
4.	If a workpiece is considered unstable on the primary datum simulator, what may be done?
5.	What is the result of applying a datum feature symbol to the width dimension on a slot?
5.	When are material boundary modifiers applicable on datum feature references?
7.	What is the difference between datum reference A-B and AB?

Name _____

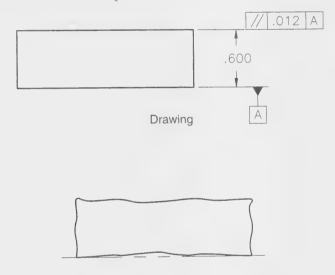
Application Problems

All application problems are to be completed using correct dimensioning techniques. Show any required calculations.

68. Identify the order of precedence for each of the datum feature references.

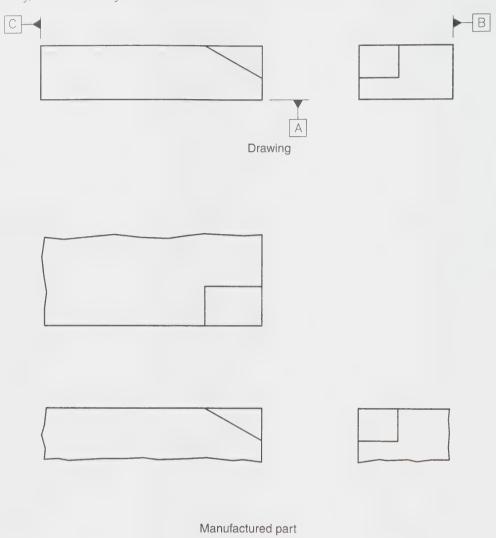


69. In the drawing, identify a datum feature reference and a datum feature. In the interpretation view, identify a datum feature and a datum plane.



Interpretation

70. A sketch of a manufactured part is given below the drawing. Sketch the planes of a datum reference frame in all the views of the manufactured part. Label each of the datum planes that make up the datum reference frame and note the number of points of contact required with the feature surface. Complete the sketch assuming that a feature control frame references datum feature A primary, B secondary, and C tertiary.



71. Complete the datum target symbol for target location A3. It is a target area with a .50" diameter.

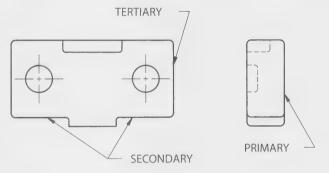


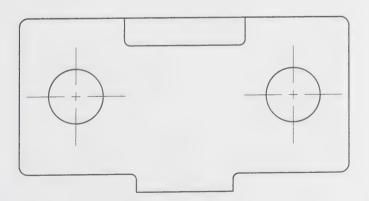
72. Identify the diameter of the given cylinder as datum feature A so that a datum axis is established.



73. Show datum target points on the given part. Use a number of targets that permit datum references in the order of precedence shown in the given drawing. Label all targets. Use basic dimensions to define target locations.

.)







- 74. Identify each of the given target symbols.





- 75. Complete the given feature control frame. Reference datum feature D primary, B secondary, and C tertiary.

\oplus	Ø	.01	5	(M)		
4	_~	• • •	_		 	

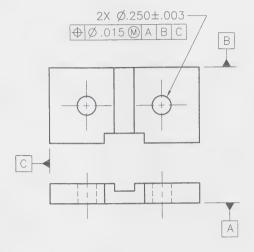
76. Identify the shaft diameter as datum feature A and the right end as datum feature B.

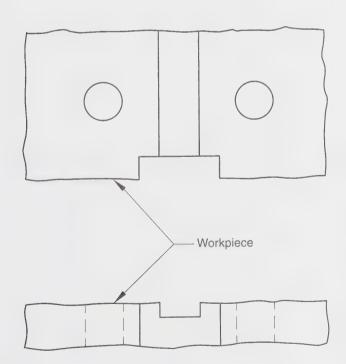


Name

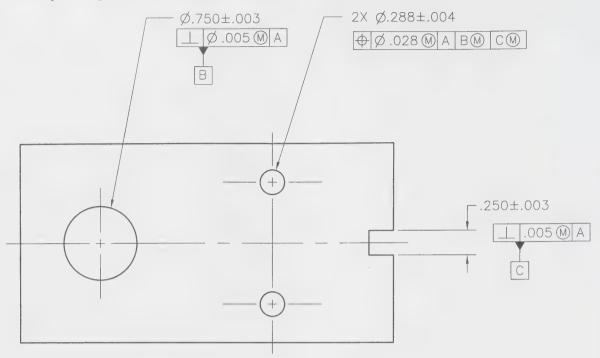
. ;

77. A drawing and manufactured part are given. Sketch a tool that properly locates the datum reference frame for the manufactured part. Show possible points of contact that would stabilize the part in the tool and meet the order of precedence for the referenced datums (Hint: Show at least three points on the primary datum feature).





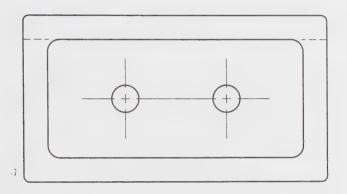
78. Sketch and dimension the gage features required to establish the datum reference frame for the shown part. Superimpose the gage on the given views.

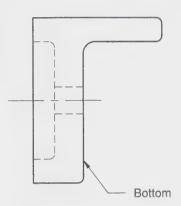




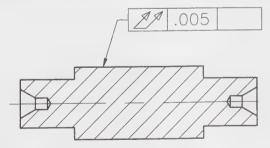
Name

79. Assume the bottom surface of the shown part is referenced in two feature control frames. It is referenced as primary datum A in one specification. It is referenced as secondary datum E in another specification. Show targets that permit the two datum references.

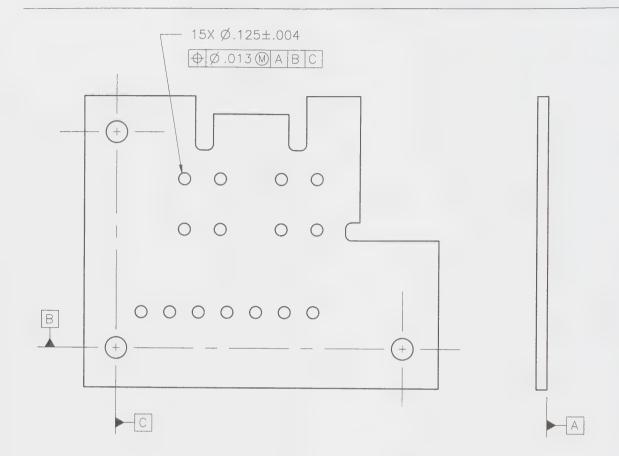




80. Identify the centerdrill countersinks as datum features A and B. Complete the total runout specification by showing a primary datum reference to compound datum features A and B.

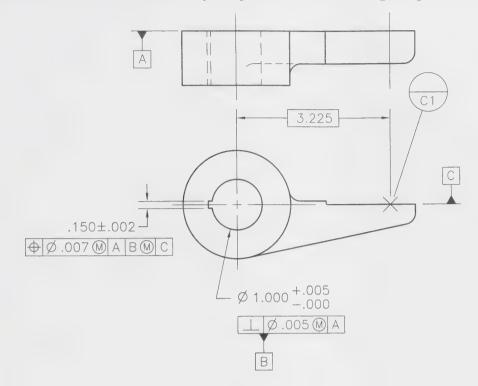


81. Explain why the shown drawing is wrong and correct the drawing.

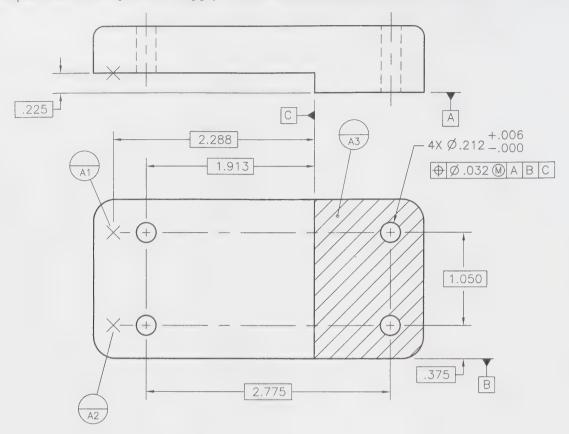


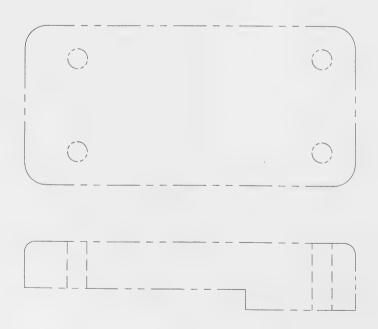
. ;

82. Sketch the datum simulators required for the given part. Apply nominal size and location dimensions for the simulators. Do not superimpose the sketch on the given part.

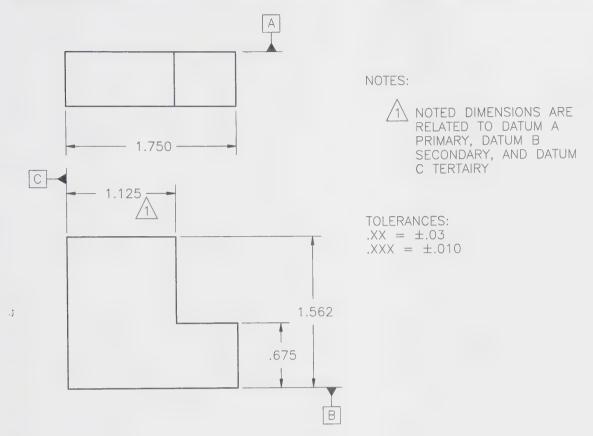


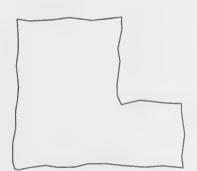
- 78
 - 83. The given part drawing shows a front and bottom view with dimensions. A front and top view are shown in phantom lines. Sketch the datum simulators required for the given part in the views where the part is shown in phantom. Apply nominal location dimensions for the target point locators.





84. Complete the interpretation drawing for the one dimension that has note #1 applied to it. Show the datums and the dimensions to the tolerance zone for the one dimensioned feature that is affected.





80 GD&T: Application and Interpretation Study Guide
NOTES

Chapter 7

Orientation Tolerances

Name Washer Det Marchall Date	24/2	Class
Reading		
Read Chapter 7 of the GD&T: Application and Interpretation exercises.	on textbook prior to o	completing the review

Objectives

A combination of activities is required to achieve the following objectives. Completing the reading assignment and the following review exercises is an important part of achieving the objectives. Familiarization with the objectives prior to completion of the reading assignment and review exercises will make mastery of the objectives easier. After completing the reading assignment and completing the review exercises, you will be able to:

- Identify, apply, and interpret orientation tolerances.
- ▼ Complete orientation tolerance specifications including one or two datum feature references.
- Explain the effects of material condition modifiers when orientation tolerances are applied to features of size.
- ▼ Calculate the virtual condition for internal and external features of size to which an orientation tolerance is applied.
- ▼ Complete tolerance specifications that include orientation and form requirements on a single feature.

Review Exercises

Place your answers in the spaces provided. Show all calculations for problems that require mathematical solutions.

Multiple Choice 1. There must be ____ datum feature reference(s) in a perpendicularity tolerance specification. A. no B. only one One or more D. two or more

82

B. both the surface to which it is applied and the opposite end of the part

C. the center plane of the toleranced feature of size

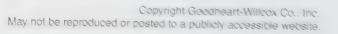
D. None of the above.

8. Aperpendicularity tolerance applied to the width dimension on a slot establishes a requirement on _____ to a value equal to the tolerance value. (A) both sides of the slot

B. the side of the slot closest to the tolerance specification

C. the center plane of the slot

D. All of the above.



Name		
A	9.	An orientation tolerance applied to may result in surface variation that lies outside the tolerance zone, but a plane tangent to the surface must be within the tolerance zone. A an individual feature B. multiple features C. a unit area D. a tangent plane
True/False		
	10.	<i>True or False?</i> Parallelism tolerances may only be applied to flat surfaces.
	11.	<i>True or False?</i> An orientation tolerance may be used to establish a location requirement.
	12.	<i>True or False?</i> An orientation tolerance should not be applied to a feature that already has another tolerance type such as a position tolerance.
.;	13.	<i>True or False?</i> An orientation tolerance including the MMC modifier and applied to an internal feature of size, such as a hole, creates a virtual condition that is smaller than the MMC size of the toleranced feature.
	14.	<i>True or False?</i> A parallelism tolerance establishes an orientation requirement, and does not establish the maximum and minimum limits of size for a feature.
	15.	<i>True or False?</i> A parallelism tolerance of .008" can define the allowable variation on the distance (location or size) between two flat surfaces.
	16.	<i>True or False?</i> A diameter symbol is needed when a parallelism tolerance is applied to control the parallelism of the axis for one hole to the axis of another hole.
	17.	<i>True or False?</i> Ninety degree angles in an orthographic view do not require dimensions to show the angle.
	18.	<i>True or False?</i> A perpendicularity tolerance must never reference two datum features.
	19.	<i>True or False?</i> A secondary datum feature reference in a perpendicularity tolerance specification stops rotation of the part on the primary datum.
Fill in the Blank		
orientration	20.	tolerances are used to specify angularity, parallelism, and perpendicularity requirements relative to one or more datums.
Parallelpin	21.	tolerance provides control of a flat surface at any angle.

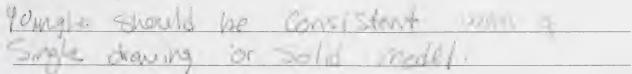
	(LFS	22.	When no material condition modifier is shown on an orientation tolerance, the material condition modifier is assumed to apply.
	NOIL	23.	A parallelism tolerance value applied to a flat surface must not be than the tolerance value that locates the surface.
	90 degre	24.	The primary datum feature referenced in a perpendicularity tolerance specification must be at a(n) angle to the toleranced feature.
	greater	25.	A produced surface with an orientation tolerance applied to it may have form variation that is equal to or than the orientation tolerance.
(M)	every fuller	26.	
	Parice	27.	
Sho	ort Answer		
	When is a material co	ondit	ion modifier applicable to an orientation tolerance?
			3/1/3
30.		hil	the term <i>virtual condition</i> when the term is associated with a shaft.
31.			riation may exist when the size dimension between two surfaces is ±.015"?
32.	Explain why it is postolerance of .010" bet	sible ween	to have a location tolerance of .050" between two holes and a parallelism the same two holes.

GD&T: Application and Interpretation Study Guide

84

Name

33. When is a 90° angle understood to be basic?



34. Determine the virtual condition for a .563", plus .005", minus .000" diameter pin that has a .012" diameter perpendicularity tolerance.

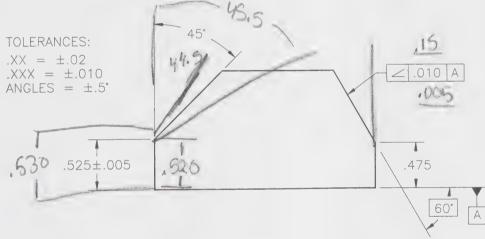
35. Determine the virtual condition for a .750", plus .006", minus .002" diameter hole that has a .010" diameter perpendicularity tolerance.



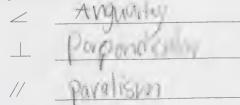
Application Problems

All application problems are to be completed using correct dimensioning techniques. Show any required calculations.

36. Show the tolerance zone for each of the inclined surfaces.



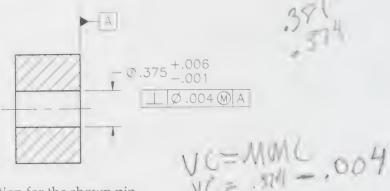
37. Identify each of the shown symbols.



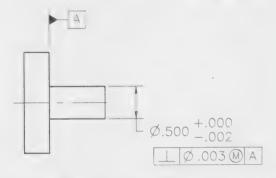
38. Complete a feature control frame that controls a flat surface to be perpendicular to datum surface A within a zone that is .006" wide.



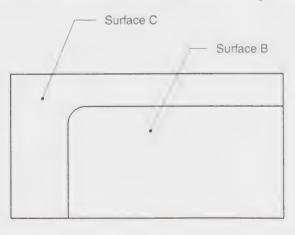
39. Calculate the virtual condition for the shown hole.

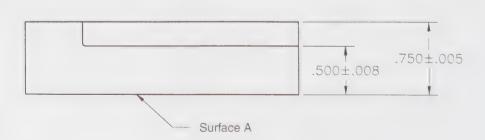


40. Calculate the virtual condition for the shown pin.

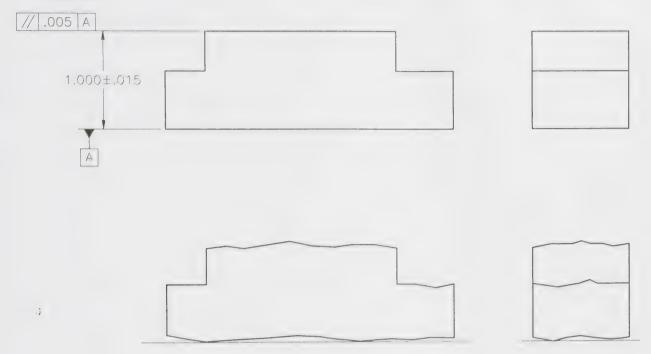


41. Surface B must be parallel within .005" to a datum established by surface A. Surface C must be parallel within .010" to the same datum. Show all required tolerance specifications.

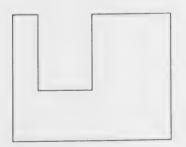




42. Complete the interpretation drawing and show the allowable tolerance zones for all specified tolerances.

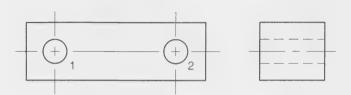


43. Apply a size dimension and tolerance to permit the slot width to vary by .020" total and apply geometric tolerance(s) to require the sides of the slot to be parallel to one another within .008".

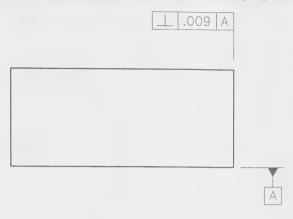


44. Apply a location tolerance of $\pm .025''$ between the shown holes. Establish one hole as a datum feature. Apply a tolerance that defines a parallelism requirement between the holes to .010'' when both holes are at MMC. There is more than one acceptable solution.

Hole #1 = Datum hole Hole #2 = Controlled hole Hole diameter = .250±.003



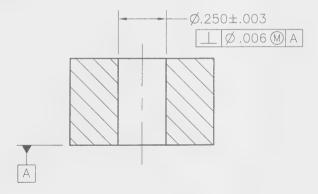
45. Complete an interpretation drawing that shows the permitted perpendicularity tolerance zone.

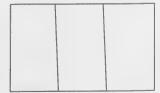


į



46. Complete an interpretation drawing that shows the permitted perpendicularity tolerance zone.



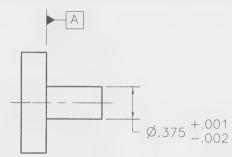


47. A hole size specification and perpendicularity tolerance is shown. Complete the given table to show each permitted hole size and show the corresponding allowable perpendicularity tolerances.

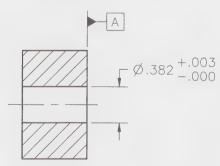
Given hole specification	Produced hole diameter	Allowable perpendicular
d + 004		tolerance
Ø.375 +.004 001	.374	,507
⊥ Ø.007 M A	.375	118
	.376	117
	.377	- 10
	378	101

.379

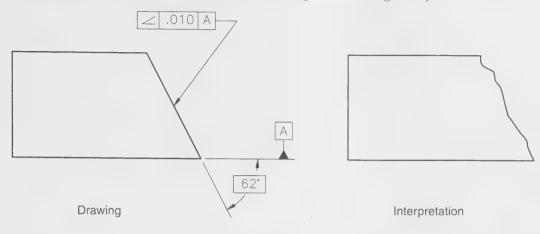
48. Apply a perpendicularity tolerance that results in a virtual condition of .379" diameter for the pin.



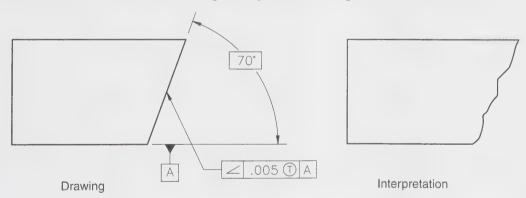
49. Apply a perpendicularity tolerance that results in a virtual condition of .379" diameter for the hole.



50. Complete an interpretation drawing that shows the permitted angularity tolerance zone.



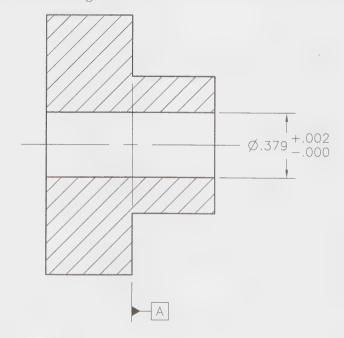
51. Complete an interpretation drawing that shows the permitted angularity tolerance zone. Show a permissible surface condition that lies partially outside the specified tolerance zone.



52. Complete a feature control frame that controls parallelism of the top surface to .015" relative to datum feature A and flatness to .005".



53. Complete a feature control frame that controls perpendicularity of the hole to .012"at MMC relative to datum feature A and axis straightness to .004" at MMC.



Chapter 8

Position Tolerancing Fundamentals

Name	Chaser	1 lee	Date	Class			
Read	Reading						
Re	_	GD&T: Application and	Interpretation text	book prior to completing the review			
Obje	ctives						
assignr Familia will ma review Con cone Ske Pro Cale Cale Use Cite	ment and the following arization with the obtained arization with the obtained are mastery of the old exercises, you will be applete feature control dition modifiers, and the proper location of the proper location of the examples that provide examples that proper to the examples that provide the allowable of the calculation technique calculation techniques.	ing review exercises is a pjectives prior to complet objectives easier. After cope able to: of frames for position to didatum references. on and shape for position and shape for position to the prove the validity of the ances for simple fixed as bonus tolerance for a prove to verify whether proves to verify whether provesses to verify whether the verify whether provesses to verify whether provesses to verify whether	an important part etion of the readir ompleting the readir ompleting the readir ompleting the readir on tolerance zone odifier on a posit MMC concept as and floating fasten roduced part on vocations of the content of	ion tolerance. s it applies to position tolerances.			
calcula	ace your answers in tions for problems the	the spaces provided. A hat require mathematic	ccurately comple [,] al solutions.	te any required sketches. Show all			
		 Location dimensing the located feature A. nominal values 	re.	_ if a position tolerance is applied to			

toleranced basic 94

Specification of a position tolerance with an MMC modifier results in

other than MMC.

A. undefined

D. None of the above.

B. bonus C. reduced

_ tolerance when the feature is produced at any allowable size

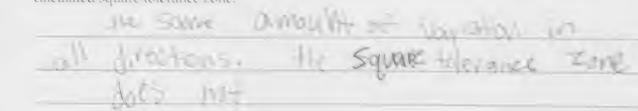
Name	
	10. The allowable position tolerance is equal to the sum of the and the bonus tolerance. A. specified tolerance B. feature size tolerance C. specified feature size D. actual produced diameter
	11. Specified hole limits of .384" MIN and .394" MAX are given. A position tolerance of .009" diameter at MMC is specified for the hole. What is the allowable position tolerance for a hole produced at .386" diameter? A007" B009" C011" D015"
	12. A position tolerance referenced to three datum planes requires that all hole locations be measured from A) the datum planes B. the datum features C. one another D. with a coordinate measurement machine
	13shaped position tolerance zones permit the same amount of hole location variation in all directions. A) Round B. Square C. Rectangular D. None of the above.
	14. A position tolerance applied to a thread defines the location requirement for the diameter when no additional notation is provided. A major B. pitch C. minor D. root
	15. A tolerance zone lies outside the toleranced feature. A projected B. position C. runout D. bonus
	16feature control frames may be applied to create a bidirectional tolerance on a slotted hole. A. Two B. Composite C. Combined D. None of the above.

96

	 A position tolerance applied to establish the location tolerance for a slot, such as a keyseat, requires that of the slot be located within the allowable tolerance. A. one side B. one end C. the center plane D. All of the above.
True/False	
	18. <i>True or False?</i> Position tolerances are applied to features of size and bounded features.
1	19. <i>True or False</i> ? Every position tolerance specification must include a material condition modifier symbol on the tolerance value.
1	20. <i>True or False?</i> Beginning with ANSI Y14.5M-1982, implied datums are no permitted on position tolerance specifications.
	21. <i>True or False?</i> It is acceptable to show a material boundary modifier on a datum reference in a position tolerance specification if the datum feature is a feature of size.
	22. <i>True or False?</i> The theoretical true position for a hole defined by a basic dimension means there is no position tolerance allowed and the hole must be perfectly located.
	23. <i>True or False?</i> The allowable size of the position tolerance zone is dependent on the amount of hole size departure from MMC if no material condition modifier is shown in the position tolerance specification.
1	24. <i>True or False?</i> An MMC modifier on a position tolerance can permit greater freedom in how a part is produced.
	25. <i>True or False</i> ? T = H – F is a formula that may be used for a floating fastener condition in which both holes are the same size and the position tolerance applied to each hole is the same value.
	26. <i>True or False?</i> If an MMC modifier is applied to a position tolerance on a hole, the allowable position tolerance increases as the hole size is increased.
	27. <i>True or False?</i> Functional gages must be used to verify hole positions when position tolerances are specified.
\ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \	28. <i>True or False?</i> Position tolerances permit utilization of the full amount of tolerance that is functionally possible for a hole, but coordinate tolerances do not.
	29. <i>True or False?</i> Position tolerances are not appropriate or needed when the allowable variation is relatively large.

. · ·	30.	
vone and annual contracts		<i>True or False?</i> Square tolerance zones do not permit the same amount of permissible hole location variation in all directions relative to the nominal position.
	31.	<i>True or False?</i> Bonus tolerances may be utilized when coordinate tolerances are applied to hole locations.
Fill in the Blank		
Round	32.	A(n) symbol placed in front of the position tolerance value indicate the tolerance zone is round.
- True	33.	Position tolerance zones are centered on the position defined by basic dimensions.
C-CCANALE S	34.	A hole for a press fit pin would typically have a position tolerance that applies at the material condition.
OVICTING :	35.	A large amount of clearance between a hole and fastener permits position tolerance than would be possible for a small amount of clearance
Position	36.	The use of the material condition results in no allowable change in the specified tolerance regardless of the produced feature size.
57	37.	Concentric circles superimposed on a grid may be used to represent tolerance zone when paper gaging.
Direct	38.	A round tolerance zone has percent more area than a square tolerance zone if the effect of bonus tolerances is ignored.
11016(469	39.	The letter P inside a circle indicates a requirement for $a(n)$ tolerance zone.
Short Answer		
		nit tolerances on the location of features?
True		VOLT OF V
applies.		may be used to show the number of holes to which a position tolerance

3	GD&T: Application and Interpretation Study Guide
	Describe one reason why implied datums should not be used, even when working to an old issue of the standard.
	tolerance values, implied intuiting fort
13.	List the two general fastener conditions for which position tolerances may be calculated.
1-1	
1 5.	What is the formula used to calculate the position tolerance for a fixed fastener condition. Assume even distribution of the allowable tolerance for the two parts.
to.	Coordinates specified for a hole are: $X = 1.375''$ and $Y = 3.250''$. A hole is produced at $X = 1.381''$ and $Y = 3.248''$. What is the diameter of position variation from true position? Show your calculations $X = 1.381''$ and $X = 1.381'''$ and $X = 1.381'''$ and $X = 1.381'''$ and $X = 1.381''''$ and $X = 1.381'''''''$ and $X = 1.381'''''''''''''''''''''''''''''''$
47.	Explain why a functionally correct round tolerance zone has a diameter that circumscribes a calculated square tolerance zone.



48. What is the effect on the hole and counterbore when a single position tolerance specification is applied to the hole and counterbore callout?

and call out.

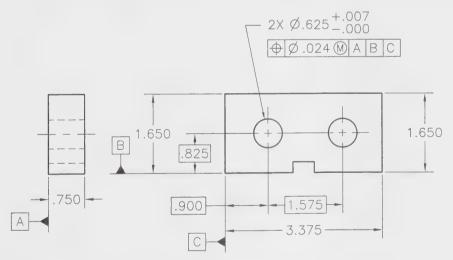
49. Explain an advantage of bidirectional position tolerances applied at MMC as compared to plus or minus location tolerances on a hole.

on a single formative

Application Problems

Åll application problems are to be completed using correct dimensioning techniques. Show any required calculations.

50. Identify a basic dimension, a datum feature symbol, and a position tolerance specification.

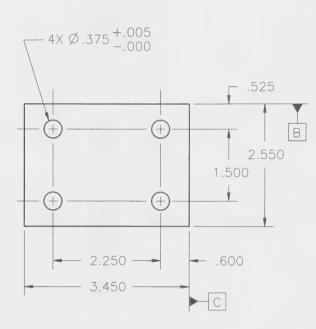


51. Complete a feature control frame for a position tolerance that is related to primary datum feature A, secondary datum feature C, and tertiary datum feature F. The tolerance zone is to be .024" diameter regardless of feature size.

52. Complete a feature control frame for a position tolerance that is related to primary datum feature D, secondary datum feature C, and tertiary datum feature G. The tolerance zone is to be .031" diameter when the feature is at maximum material condition.

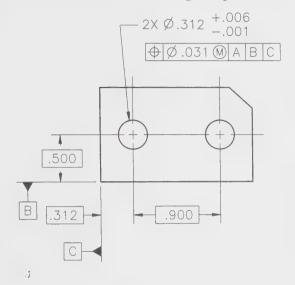
53. Draw the shown tolerance specification in an acceptable location that indicates the tolerance applies to all four holes. Make the necessary dimensions basic.

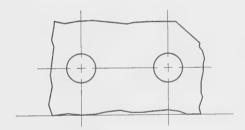
⊕ Ø.019 M A B C

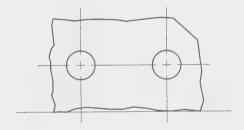


Name

54. A drawing is given and below the drawing are sketches of two produced parts with surface variations exaggerated. Assume the holes are produced exactly on the true positions defined by the drawing. Show dimensions on the produced parts to indicate how the location dimensions are measured on each of the given parts. Show any datum planes that may be needed.







55. Complete calculations to determine the allowable position tolerance for each of the applications shown in the table. Each of the applications is for a floating fastener. Insert your answers in the given table.

SPECIFIED HOLE DIA	FASTENER DIA AT MMC	ALLOWABLE POSITION TOLERANCE AT MMC
.221±.003	.190	
.219±.002	.190	
.282±.004	.250	

56. Complete the given table. All problems are for a floating fastener application.

HOLE DIA AT MMC	FASTENER DIA AT MMC	ALLOWABLE POSITION TOLERANCE AT MMC
.189	.164	
	.190	.031
.279		.029

57. Complete the given table. All problems are for a fixed fastener application.

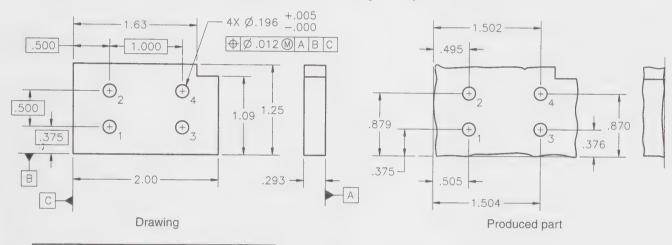
CLEARANCE HOLE DIA AT MMC	FASTENER DIA AT MMC	ALLOWABLE POSITION TOLERANCE AT MMC		
.282	.250			
.218		.014		
	.312	.021		

Name

58. Show your calculations. Enter the X and Y variations for each produced hole in the two tables provided. Calculate the amount of X and Y variation from true position for each hole and enter the variation data in the two tables. Use one or both of the following methods to determine if the produced holes are in acceptable locations.

Solution Method 1. Plot the hole locations on the given grid. Label each hole location with the hole identification number. Draw circles to represent tolerance zone diameters. Note each hole location as acceptable or unacceptable. Each grid space equals .001".

Solution Method 2. Calculate and enter in the given table the bonus tolerance for each hole. Calculate and enter in the table the allowable position tolerance for each produced hole. Determine by calculation or conversion table the diameter of position variation for each hole and enter the value in the table. Enter in the table whether to accept or reject the hole.



Hole #	.199		.201	
Diameter				
	X	Y	X	Y
Measured Location				
Drawing Dimension	.500	.375	.500	.875
Variation				

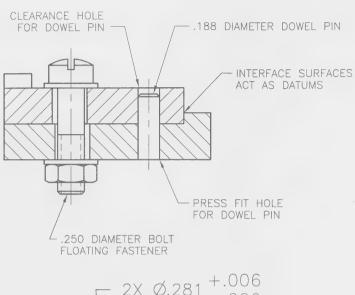
Hole #	3		4	
Diameter	.200		.200	
	X	Y	X	Y
Measured Location				
Drawing Dimension	1.500	.375	1.500	.875
Variation				

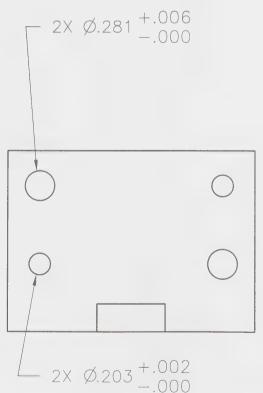
Plotted coordinate variations and position tolerance zones

Measured hole data

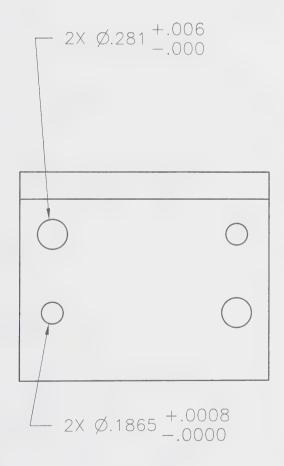
Hole #	Bonus Tolerance	Allowable Position Tolerance	Measured Position Variation	Accept or Reject
1	.003	.015	.0100	Accept
2				
3				
4				

59. Complete the detail drawings of the two given parts to the extent required to define hole location requirements. Hole sizes and fastener sizes are provided. Select and identify datums. Dimension the true positions of the holes. Calculate and apply position tolerances that ensure the two parts can be assembled. Use projected tolerance zones if needed. The limits of size for the dowel pin are .1876 to .1878 diameter and may be rounded off to .188 diameter for calculating the position tolerances. The bolt should be assumed to have an MMC size of .250 diameter.



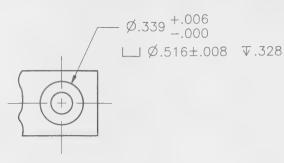




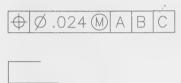




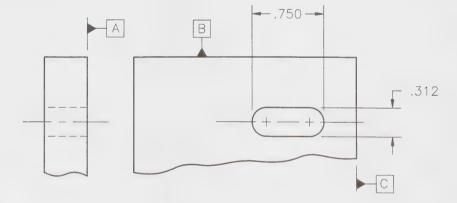
60. Complete the hole specification including a position tolerance of .018" diameter at MMC relative to primary datum feature A, secondary datum feature B, and tertiary datum feature C. Apply the tolerance specification in such a manner that the tolerance is applicable to both the hole and counterbore.



61. Redraw the feature control frame to specify a projected tolerance zone that extends .375".



62. Apply a position tolerance on the given slot to permit .045" location variation in the X axis and a .015" in the Y axis.



Chapter 9

Position Tolerancing— Expanded Principles, Symmetry, and Concentricity

Name		Date	Class
Reading			
Read Chapter 9 of the exercises.	ne GD&T: Application a	and Interpretation textboo	k prior to completing the review
Objectives			
assignment and the follor Familiarization with the will make mastery of the review exercises, you wive Explain functional gave Specify and explain of Explain the effect of uve Specify separate patter Specify position toler Specify tolerances to very Control coaxial feature	owing review exercises objectives prior to cone objectives easier. After a ble to: aging methods for check to be a ble to: aging methods for check to be a ble to: aging methods for check to be a ble to: aging methods for check to be a ble to: aging methods for check to be a ble to: aging methods for check to be a ble	is an important part of a inpletion of the reading a er completing the reading cking hole position tolera erance specifications. eature references in multi- croups of features not act	ssignment and review exercises g assignment and completing the ances specified at MMC.
		l. Show all calculations f	for problems that require
mathematical solutions.			
Multiple Choice	telerance zone A) fixed pos	es that have relativ	n tolerance specification establisheve to the referenced datums.

C. only a fixed orientationD. no orientation requirement

Name		
13	9.	Coaxial (or in-line) holes when using a position tolerance to spece a tolerance that controls the in-line condition. A. must have the same diameter may have different diameters C. must have one hole referenced as a datum D. None of the above. tolerances should only be used when it is necessary to control a derived median line RFS relative to a datum axis RMB. A. Position B. Concentricity C. Runout D. Composite position
True/False		
	_ 11.	True or False? Parts inspection may be accomplished by using functional gages to check position tolerances that are specified at MMC.
- · · · · · · · · · · · · · · · · · · ·	12.	<i>True or False?</i> In composite position tolerances, the feature-relating tolerance defines allowable feature-to-feature positions.
	13.	<i>True or False?</i> The true positions of a feature-relating tolerance zone framework must all be within the pattern-locating tolerance zones.
	14.	<i>True or False?</i> A feature-relating tolerance zone framework must be properly oriented (rotational degree of freedom constrained) relative to the primary datum that is referenced in the second line of a composite position tolerance specification.
	15.	True or False? If the first set of measurements for a pattern of holes does not meet the feature-relating tolerance specification when paper gaging different holes within the pattern may be used to establish a coordinate system for another set of measurements.
	16.	True or False? Two position tolerance symbols may be used in a two segment feature control frame to specify a composite position tolerance
	17.	<i>True or False?</i> A functional gage containing a pin that is sized to the virtual condition of a hole automatically checks the hole location and both size limits for the hole.
	18.	<i>True or False?</i> All references to datum features of size must include the maximum material boundary modifier when specifying composite position tolerances.
	19.	<i>True or False?</i> The two gages used to check the pattern-locating tolerance and the feature-relating tolerance for a pattern of holes both have the same diameter of gage pins.

	_ 20.	<i>True or False?</i> All holes are known to act as a single pattern if the holes are the same diameter.
	21.	<i>True or False?</i> A composite position tolerance, instead of concentricity, applied to two or more coaxial (in-line) holes must contain at least one datum feature reference for the feature-relating tolerance.
- i	22.	<i>True or False?</i> Position tolerances are typically applied to coaxial holes when the main concern is assembly of the parts.
	23.	<i>True or False?</i> Symmetry tolerances should not be applied to any features other than hole patterns.
	24.	<i>True or False?</i> Concentricity tolerances may be used to define the allowable surface variations of one cylinder relative to a datum axis.
Fill in the Blank		
	25.	A segment position tolerance specification made RFS requires all hole locations be within position tolerance zones that are all the same size.
	26.	In composite position tolerances, thelocating tolerance establishes the hole pattern position requirements relative to the datum references frame.
	27.	The segment of a composite position tolerance always specifies the pattern-locating tolerance.
	28.	The segment of a composite position tolerance has the same effect as a single segment position tolerance specification.
	29.	Paper gaging the feature-relating tolerance for a pattern of holes may be accomplished by using one of the holes as the for measurements within the pattern.
	_ 30.	A functional gage for verifying hole locations automatically permits utilization of any allowable bonus tolerance since gage pins are sized to the of the holes being checked.
	_ 31.	An MMB modifier on a primary datum feature reference requires the size of the datum feature be used to establish the datum location if Rule #1 is applicable to the feature.
	_ 32.	Placing the words under a position tolerance specification results in the associated group of holes acting as a separate pattern from any other holes or features.
	_ 33.	If two groups of holes are toleranced with composite position tolerances that reference different datums, patterns of features are created.
	34.	Symmetry requirements that apply at MMC are specified using the symbol.
	_ 35.	Concentricity is always specified with the tolerance applicable of feature size.

GD&T: Application and Interpretation Study Guide

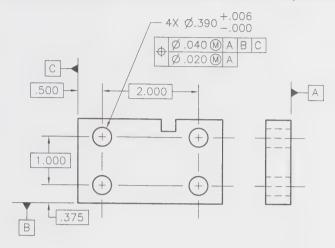
110

Chapter 9	Position Tolerancing—Expanded Principle	les, Symmetry, and Concentricity
-----------	---	----------------------------------

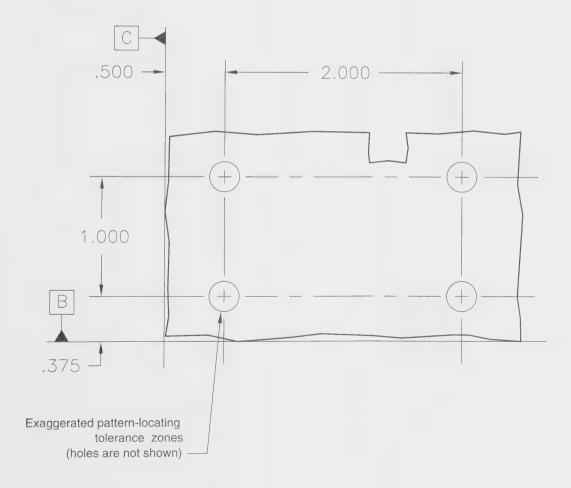
111

Nam	e
Sho	ort Answer
36.	What requirements apply to the specification of datum features in the second segment of a composite position tolerance?
37.	Explain the feature-relating tolerance zone framework requirement for a composite position tolerance specification that is applied to a pattern of holes when no datum reference is shown in the second segment.
38.	Why are two holes in a hole pattern used to establish a coordinate system when making measurements to check the feature-relating tolerances?
39.	What is a functional gage?
40.	What must be accomplished with the datum simulator if the outside diameter of a shaft is referenced as a datum feature with no modifier applied to the reference?
	p)

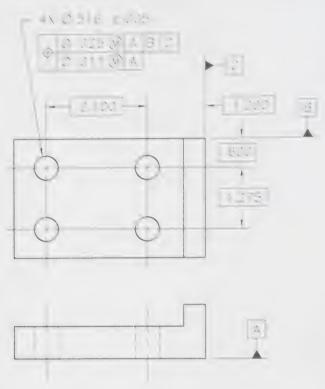
47. The pattern-locating tolerance zone framework and the pattern-locating tolerances are shown on the illustrated part. Holes are not shown. Show one possible location of the feature-relating tolerance zone framework that does not coincide with the pattern-locating tolerance zone framework. Also show the feature-relating tolerance zones. Show one permissible point for the center location of each hole.

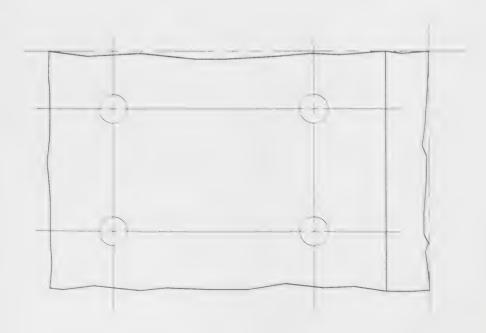


.;



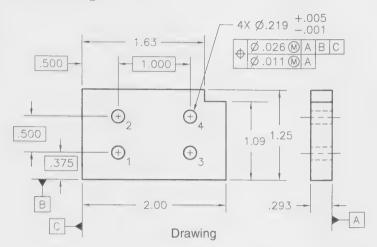
- 114
 - 48. The pattern locating tolerance zone framework and the pattern-locating tolerances are shown on the given part. Show one possible location of the feature-relating tolerance zone framework that does not coincide with the pattern-locating tolerance zone framework.





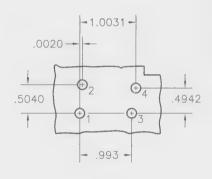
Name

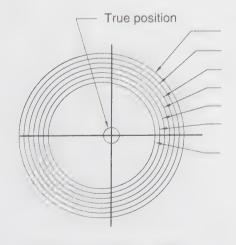
49. Complete the steps necessary to prove acceptability or rejection of the given part using paper gaging techniques. Fill in the blanks in the table. Plot the position variations on the provided grid. Label the concentric circles to indicate allowable position tolerance and corresponding hole sizes. Answer the questions in the figure.



HOLE-TO-HOLE LOCATION VARIATION

Hole #	1		2		3		4	
Diameter	.22	22	.223		.221		.223	
	X	Y	X	Y	X	Υ	X	Y
Measured Location	0	0	.0020	.5040	.9930	0	1.0031	.4942
Drawing Dimension	0	0	0	.500	1.000	0	1.000	.500
Variation								

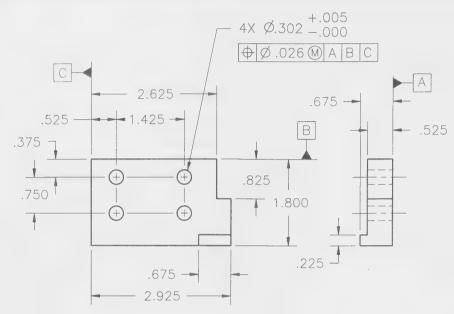


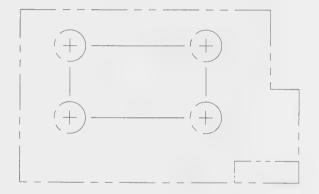


Hole-to-hole relative positions

- ____ Is the given location of the concentric circles allowed?
- _____ Is bonus tolerance required to make any of the holes acceptable?
- _____ Is the feature-relating tolerance met?

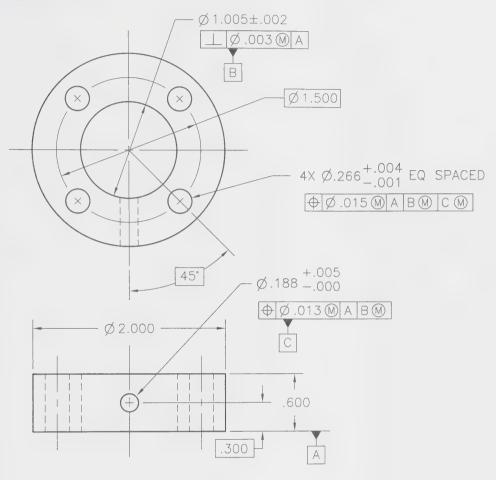
50. Design a functional gage that checks the hole positions in the given part. Do not apply gage tolerances. Superimpose the gage on the given part where the part is shown with phantom lines.



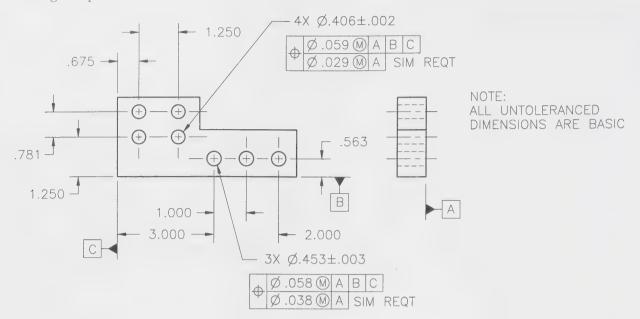




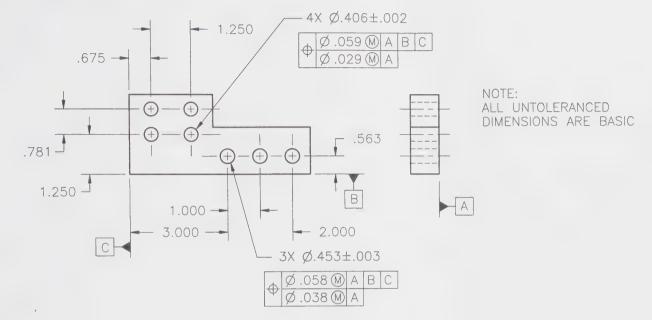
51. Calculate the diameter of a pin that establishes the secondary datum for the shown position tolerance specifications.

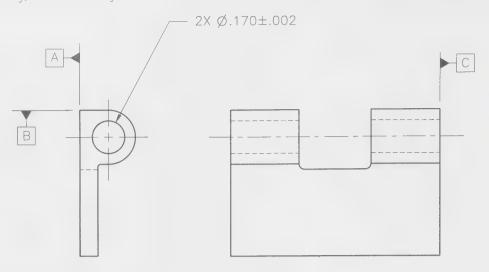


52. Complete a drawing of the gage(s) needed to verify the feature-relating tolerance for all the holes in the given part.

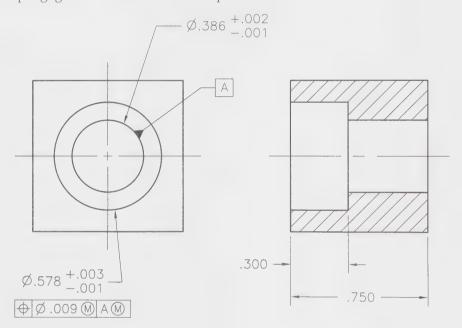


53. Complete a drawing of the gage(s) needed to verify the feature-relating tolerance for all the holes in the given part.

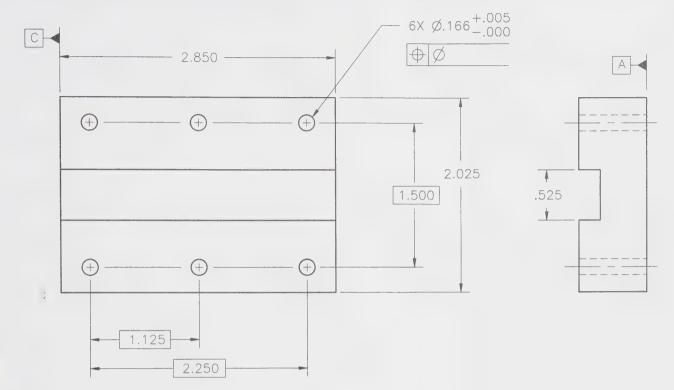




55. Sketch a simple gage that verifies the shown position tolerance.



56. Apply any additional dimensions and tolerances needed to define hole locations that are symmetrically located to the slot within a .026" diameter zone when the holes and slot are at MMC. Datum A is primary, the slot secondary, and one end of the part tertiary.



122 GD&T: Application and Interpretation Study Guide	
NOTES	

Chapter 10

Runout

Name	er)	Date	Class
Reading			
Read Chapter 10 cexercises.	of the GD&T: Application	on and Interpretation textbo	ook prior to completing the review
Objectives			
will make mastery of the review exercises, you was a Describe the two ty was Complete an interp was Apply both types of	he objectives easier. A will be able to: rpes of runout tolerance retation drawing show frunout tolerances on rances using multiple oplication for a runout	ter completing the reading. Tes. Ving how each of the runce circular features and face datum feature references.	
mathematical solutions		led. Show all calculations	for problems that require
Multiple Choice	A. Cylind B. Total C. Face su D. Circula	urface ur nout may be measured on	cross an entire surface. n any that has circular

flat surface All of the above.

124 GD&T: Application and I	nterpretation Study Guide
124 GD&T: Application and I	3. A circular runout symbol has arrows. A. one B. two C. either one or two D. None of the above. 4. The material condition that always applies to runout tolerances is A. MMC B. LMC C. RFS D. All of the above. 5. Runout tolerance specifications must include a A. datum feature reference B. MMC or LMC modifier C. three place decimal tolerance value D. None of the above. 6. Datum reference B-C indicates A. one datum created by two datum features B. two datums created by two datum features C. a primary and secondary datum D. a single datum created by one datum feature that is identified with the letters B and C 7. A(n) line may be used to indicate a limited area of application for a tolerance specification.
True/False	A. object B. center C. phantom D. chain
nue/raise	
	8. True or False? Runout may only occur on a cylindrical surface.
Janes .	9. True or False? One runout reading taken at a cross section on a 3.00" long shaft is adequate to verify a circular runout specification for the 3.00" shaft.
\ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \	10. <i>True or False?</i> Runout tolerances applied to internal features require notations to explain what the specification means.
	11. True or False? One datum reference is all that is ever needed for any runout tolerance specification.
	12. <i>True or False?</i> A runout tolerance may not exceed the size tolerance on the controlled feature.

Name	
Fill in the Blank	
Conface 1	 Runout is the amount of variation that is allowed relative to an axis of rotation.
1	4. When using a dial indicator for inspection of runout, the part must be on an axis to make the runout measurements.
DM/20010 1	5. Two features acting together to establish a single datum axis, such as A–B, through those features is referred to as datum features.
Dail Mine	6. Runout tolerances applied to the outside diameter of a gear blank are measured by rotating the workpiece on the datum axis with a against the outside diameter of the gear blank.
ton item	7. When both a primary and secondary datum reference are shown in a runout tolerance specification, usually the datum features include one surface and one face (flat) surface.
1	8 runout is the variation across an entire surface relative to an axis of rotation.
Short Answer	
19. Explain how a circular r indicator.	runout requirement is checked on a cylindrical feature when using a dial
one mely	d Colory March March 1911
in Conte	Rotated on mais will be
20. Why is a diameter symb	ool not used in runout tolerance specifications?
Can Parle	The My symbol is double and
21. What is achieved by the the axis of rotation?	application of a total runout tolerance on a surface that is perpendicular to
ye dill	MISONA IS MADURA MILIOSE THE YEAR
Swface	dirtection perpendicular to avis

22	Give one reason	why there	might be a d	latum referenc	e such as D-	-E in a runout	tolerance
	CITY COILC I COLDOIT	. TTLLY CLICAC	11119110 20 01 0	TOTO CELLE I CICI CITE	e Deterr bro as		

A-B day velerce

23. How may a face surface, as a secondary datum reference, be beneficial when a runout tolerance is referenced to a primary datum axis?

Secondary with relieve be penetral

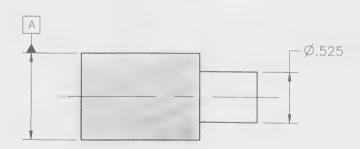
24. List two geometric shapes other than a cylinder or flat surface that may be controlled with circular runout.

runout his any surpra with Circular content

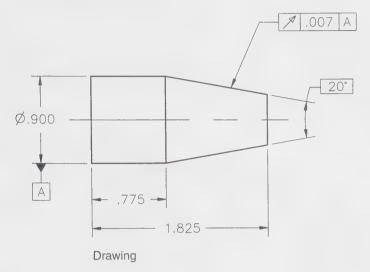
Application Problems

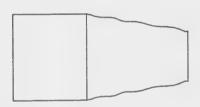
All application problems are to be completed using correct dimensioning techniques. Show any required calculations.

25. Show two ways to apply a circular runout tolerance specification of .006" on the small diameter relative to datum axis A.



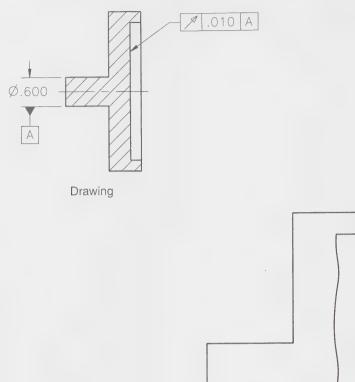
26. Sketch a setup and measurement method that may be used to check the runout tolerance. Also show the acceptable tolerance zone at multiple locations on the feature.



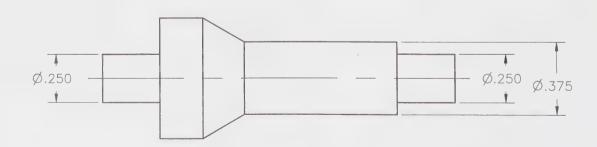


27. Complete a feature control frame that specifies a circular runout tolerance of .008" relative to an axis established by datum feature C.

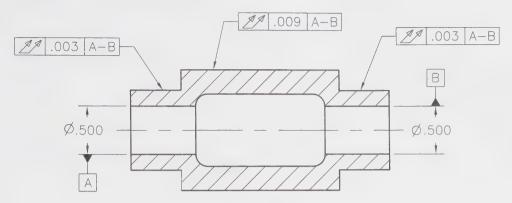
- GD&T: Application and Interpretation Study Guide
 - 28. Sketch a setup and measurement method that may be used to check the runout tolerance. Also show the acceptable tolerance zone at multiple locations on the feature.

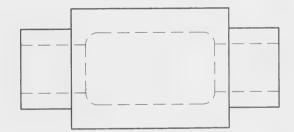


29. Apply the necessary symbology to control the circular runout of the .375" diameter to a value of .006" relative to an axis established by the two .250" diameter bearing surfaces.

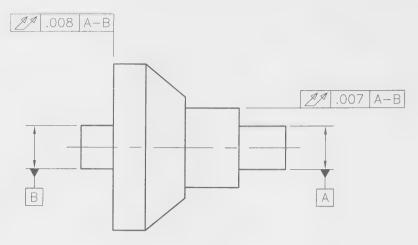


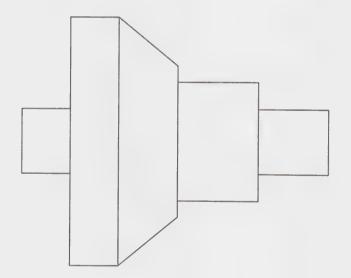
30. Sketch a setup and measurement method that may be used to check the runout tolerances. Also show the acceptable tolerance measurements.





31. Sketch a setup and measurement method that may be used to check the runout tolerance. Also show the acceptable tolerance measurements.





Chapter 11

Profile

Name(/ V // / / / / /	Date	Class					
Reading	Reading						
Read Chapter 11 of the GD&T: App exercises.	olication and Interpretation textbool	k prior to completing the review					
Objectives							
A combination of activities is requ	ired to achieve the following obje	ectives. Completing the reading					

- Define line and surface profile tolerances.
- ▼ Apply profile tolerances to define allowable variation within a limited zone on a feature or for all of a feature.

assignment and the following review exercises is an important part of achieving the objectives. Familiarization with the objectives prior to completion of the reading assignment and completing the review exercises after reading will make mastery of the objectives easier. After completing the reading

- Apply profile tolerances to extend all around the profile shown in a drawing view.
- Complete profile tolerance specifications to achieve any of the possible levels of control.
- Sketch the tolerance zone created by profile tolerance specifications.

assignment and completing the review exercises, you will be able to:

- Specify coplanarity requirements using profile tolerances.
- Identify profile tolerances as the means for specifying allowable variation for conical surface form, orientation, and location.
- Draw a composite profile tolerance specification.

Review Exercises

Multiple Choice

Place your answers in the spaces provided. Show all calculations for problems that require mathematical solutions.

	Only the is different between the format of a line profile and surface profile feature control frame.
	A. datum referencing method

- B. use of basic dimensions
- all around symbol usage
 D. tolerance symbol

Name	
10.	A profile tolerance may be specified not to extend across an entire feature by indicating a A limited extent of application drawing the tolerance zone C. drawing a line to one side of the basic profile D. None of the above. A surface profile tolerance applied to a cone should include to establish location, orientation, form, and size requirements. A. no datum feature references B. no basic location dimensions neither datum feature references or basic location dimensions datum feature references and basic location dimensions
True/False	
12.	<i>True or False?</i> Profile tolerances are always specified with the MMC modifier.
	<i>True or False?</i> A curved surface must be defined by basic dimensions when a profile tolerance is applied to the surface.
14.	<i>True or False?</i> Surface profile may only be used to control the form of a curved surface.
15.	<i>True or False?</i> Even when an all around symbol is used, profile tolerances do not extend past abrupt changes in direction.
16.	<i>True or False?</i> There is no number shown following the unequally disposed symbol if the entire profile tolerance zone goes inside the material relative to the basic profile of the surface.
17.	<i>True or False?</i> When used, unequally disposed profile tolerances must be applied to permit a plus size tolerance rather than a minus size tolerance.
18.•	<i>True or False?</i> A feature controlled by a profile tolerance may be located by a basic dimension and when basic location dimensions are shown the profile tolerance must include appropriate datum feature references.
19.	<i>True or False?</i> A composite profile tolerance may be used to specify a small tolerance for form of a surface and a large tolerance for the form, orientation, and location relative to one or more datums.
20.	<i>True or False?</i> One method of specifying coplanarity of multiple flat surfaces is to apply a flatness tolerance.

Nama		1	16 18	20		
Name	_1/	1	1		 	

31.	Explain the impact of applying a basic dimension for the location of a surface that has a profile
	tolerance including datum feature references in the feature control frame.

Greate a tollerate of profile Marian Page Universe of

32. Place an *X* by each characteristic that affects the required level of control on a feature.

Line or surface profile symbol

____ Datum feature references

_____Total area of the controlled surface

Basic location dimensions

____Curved or flat surface

33. How can a coplanarity requirement for multiple flat surfaces be specified?

the sport are contplied

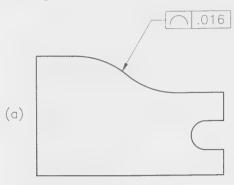
Application Problems

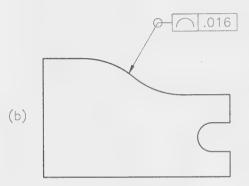
All application problems are to be completed using correct dimensioning techniques. Show any required calculations.

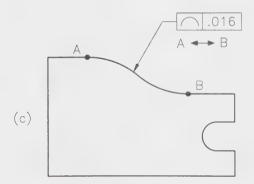
34. Apply a line profile tolerance that only controls the form of the curved surface within a boundary .025" wide.



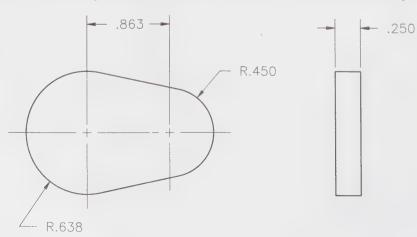
35. Show the tolerance zone created by each of the given tolerance specifications. Superimpose the tolerance zone on the given drawing.





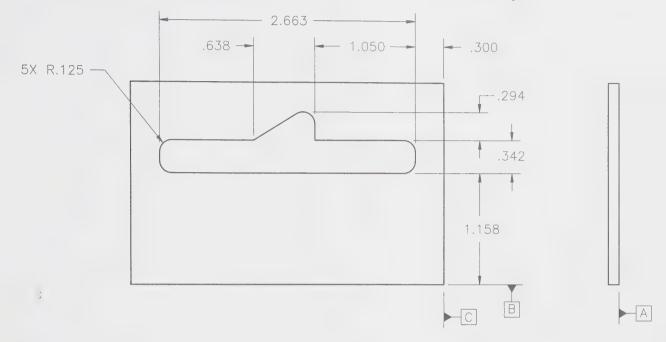


36. Complete the drawing to the extent necessary to control the surface profile all around the perimeter of the part within a boundary .040" wide. Indicate basic dimensions where they are needed.

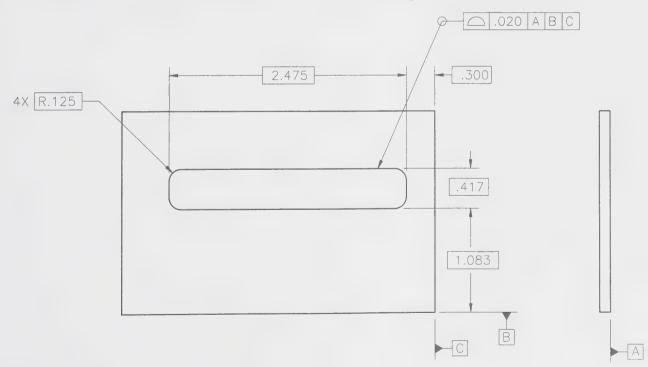


Name

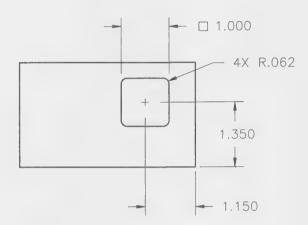
37. Control the surface profile of the given slot all around within an unequally disposed zone .015" to the inside of the material (reduces the material). Also, control both location and orientation of the profile tolerance zone to three datums. Indicate basic dimensions where they are needed.



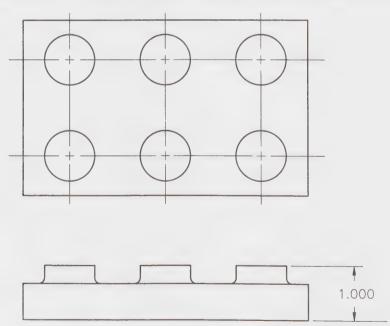
38. Show the tolerance zone for the given slot. Superimpose the tolerance zone on the given drawing. Dimension the width of the slot and the offset from the true profile.



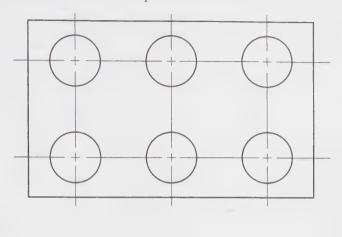
39. Control the form and size of the punched hole within a surface profile of .010". Use position at MMC to specify a location tolerance of .020". Identify and use datum features as needed. Indicate basic dimensions as needed.



40. Require flat and coplanar bosses within an .008" tolerance zone. Allow location and parallelism within \pm .015" relative to the bottom surface. Use composite profile to specify the requirements.

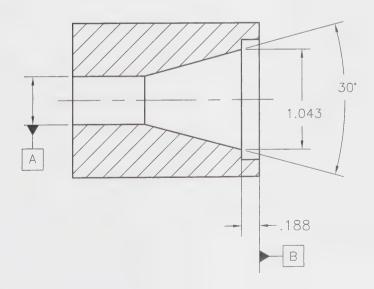


41. Require flat and coplanar bosses that are located within an .008" tolerance zone. Require the zone to be centered 1.000" from datum A and parallel to datum A.



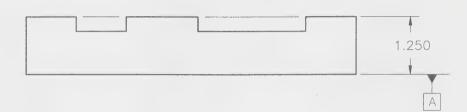


42. Specify a tolerance zone that controls the cone surface size, form, orientation, and location relative to datum axis A and datum plane B within a boundary that is .018".



43. Apply a composite profile tolerance to control coplanarity to within .005" and location and parallelism to datum A within .025". Apply basic dimensions where needed.





Chapter 12

Practical Applications and Calculation Methods

Name		Date	Class
Reading			
Read Chapter 12 of exercises.	f the GD&T: Applica	tion and Interpretation textbo	ook prior to completing the review
Objectives			
assignment and the foll Familiarization with the will make mastery of th review exercises, you w ▼ Calculate position to fastener application. ▼ Distribute the total a ▼ Specify projected tol ▼ Determine the amou ▼ Properly use zero po ▼ Apply paper gaging	owing review exerce objectives prior to be objectives easier. Fill be able to: olderances when more vailable position tolderance zones for fix ant of tolerances at techniques to determine to tolerance efficient of tolerance efficient tolerance efficient of tolerance efficient	cises is an important part of a completion of the reading a After completing the reading the than two parts are stacked	assignment and review exercises g assignment and completing the d in a floating fastener or fixed which position tolerances are applied ent interference conditions. ably. uring freedom. eets drawing requirements.
Place your answers mathematical solutions		rided. Show all calculations f	for problems that require
Multiple Choice			
	the edges A. origir B. datur	are referenced as in th	fastener condition must align, then ne tolerance specification.

D. mated surfaces

GD&T: Application and Interpretation Study Guide

142

Name	
	 10. A hole size specification of .210" minimum and .216" maximum diameter has a position tolerance specification of .020" diameter MMC. A .190" diameter floating fastener passes through the hole. If the hole is produced at .208" diameter and has a position variation of .012" diameter, what should be done? A. Accept the part since it meets the specification. B. Accept the part since it is functional. C. Reject the part and throw it away. D. Rework the part to make the hole an acceptable diameter.
	 11. For a floating fastener application, a hole size specification of .385" minimum and .395" maximum diameter has a position tolerance specification of .010" diameter MMC. If the position tolerance is changed to .000" diameter MMC, a minimum hole diameter of inch must be specified with the maximum size limit remaining .395". A375 B380 C385 D390
}	 12. Concentric circles used to paper gage a feature-relating tolerance requirement relative to the graph origin. A. must be centered B. are free to float C. are offset a distance equal to the location of the nearest hole D. None of the above.
	 13. If a single segment tolerance specification is applied to a single flat surface and does not include any datum references, the tolerance is either A. form or runout B. form or orientation C. form or profile D. profile or orientation
	 14. A single feature may require a maximum oflevel(s) of control, each specified in a separate feature control frame. A. no B. one C. two D. None of the above.
	 15. A flat surface may have a perpendicularity tolerance of .017" applied to it and also have a tolerance of .008" applied to further refine the surface form. A. flatness B. parallelism C. position D. circularity

29. Evenly distributed position tolerances for a fixed fastener condition are

calculated using what formula?

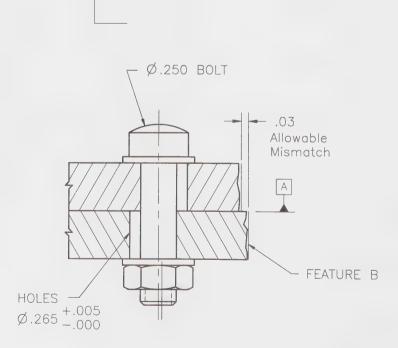
Nam	e USIV)	
	2	_ 30.	The total available position tolerance for a fixed fastener condition may be distributed between two parts using what formula?
1)(Ojected	_ 31.	A tolerance zone controls the location outside of the feature.
		_ 32.	A correctly specified zero position tolerance at results in all functionally good parts being acceptable.
		_ 33.	A specified hole diameter of .163" minimum and .168" maximum has a specified position tolerance of .025" diameter at LMC. A produced hole of .165" diameter has an allowable position tolerance of inch diameter
Shc	ort Answer		
34.	If three stacked parholes calculated?	ts all h	ave the same diameter clearance holes, how are position tolerances for the
35.;			on tolerance for a fixed fastener application is .022", what would be wrong eter tolerance on one part and .002" diameter tolerance on the other part?
36.	Describe a fixed fas	tener c	ondition.
37.	Why is the manufaction fixed fastener conditions		process considered when distributing tolerances between two parts in a

with a]	position tolerance of .	oolt is specified to have .010" diameter at MMC .aximize manufacturin	C. What may be done	nd .268" maximum diame e to the hole size and

Application Problems

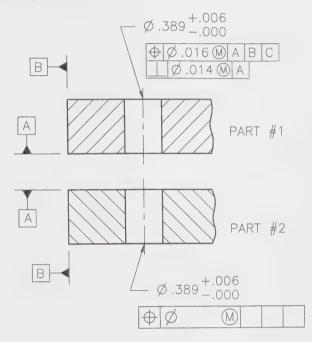
All application problems are to be completed using correct dimensioning techniques. Show any required calculations.

40. Complete a composite position tolerance that may be applied to the pattern of holes in each part. Bolts measuring .250" diameter pass through the holes. The datum features on the part may be misaligned by .030".

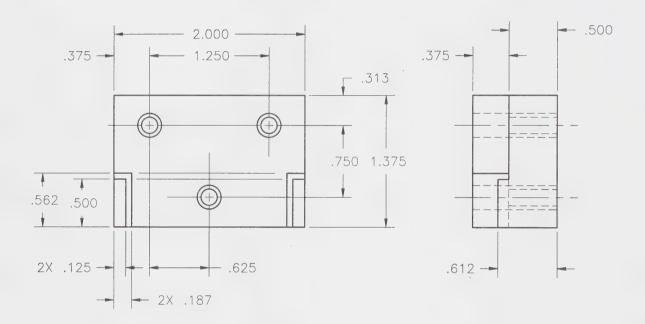


Name _____

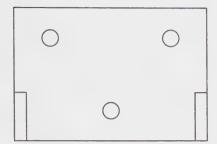
41. Apply the maximum allowable position tolerance specification on the untoleranced hole. A .375" diameter bolt passes through the holes.



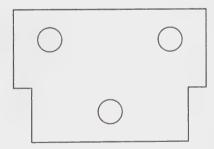
42. An assembly drawing is given. Draw one view of each part that shows the hole patterns. Dimension the hole pattern and apply tolerances for a fixed fastener condition with a .250" diameter bolt and clearance holes .292" at MMC.



Name _____

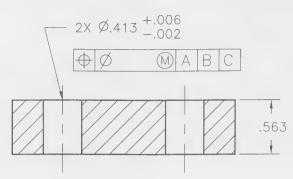


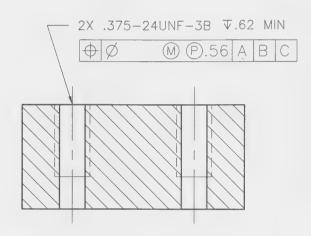




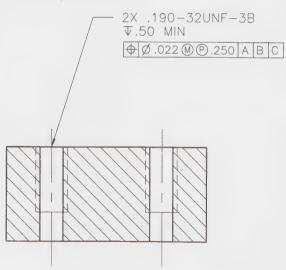


43. Calculate and apply position tolerances for the two given parts. Apply 66% of the total tolerance on the threaded holes.





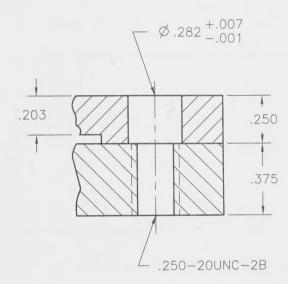
44. Show the tolerance zones for the given holes.



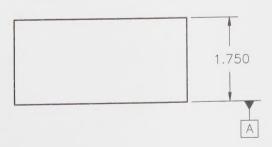
Name ____

:;

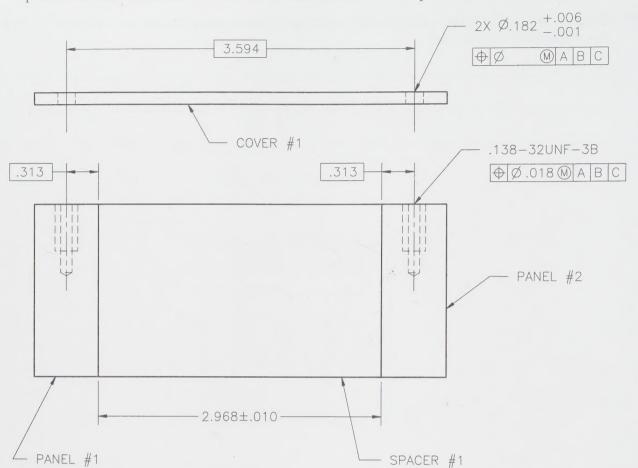
45. What is the correct projected distance for a position tolerance applied to the threaded hole? Why is that distance the correct one?



46. Apply a size tolerance of $\pm .030$ " for the given dimension. Require the top surface to be parallel to datum A within .020" and flat within .009".

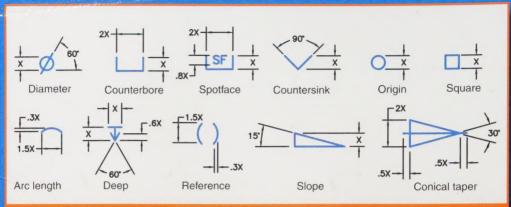


47. Calculate the allowable specified position tolerance for the specified clearance hole on the shown plate. Assume datums are selected to minimize tolerance stackup.

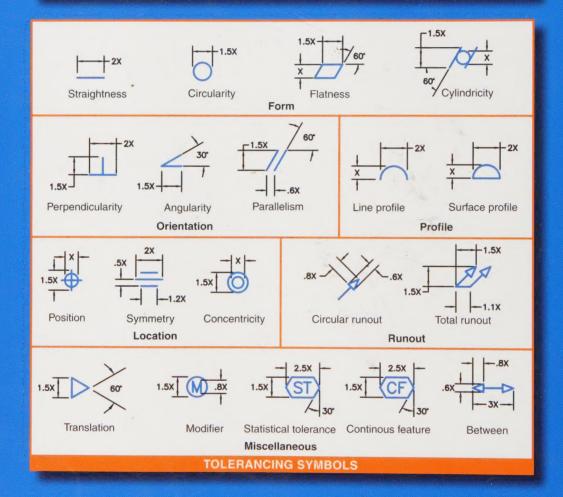


:)

10



DIMENSIONING SYMBOLS





9||781631||261152|| 9999